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STRESSES AND DEFORMATIONS IN WINGS

SUBJECTED TO TORSION

By B. F. Ruffner and Eloise Hout

Oregon State College



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SUMMARY

Basic equations of Karman and Chien (given in "Torsion with Variable Twist," Jour. Aero. Sci., vol. 13, no. 10, Oct. 1946, pp. 503-510) are solved by representing the shape of a torsion box by means of a Fourier series. The coefficients of the series are determined by conventional methods. Angles of twist, longitudinal stresses, and shear stresses are determined in terms of the series coefficients. The method is applied to the calculation of angles of twist and stresses in torsion boxes of rectangular, elliptical, and airfoil cross section.

Results obtained for angles of twist and normal stresses are in good agreement with results of Karman and Chien except at sharp corners. Results obtained for shear stresses indicate the necessity for use of a large number of terms of the series for satisfactory accuracy.

INTRODUCTION

Experimental and theoretical investigations related to torsional stresses and deformations are given in references 1 to 18. In reference 1, Karman and Chien have developed equations applicable to the problem of restrained torsion of tubes of arbitrary constant cross section loaded with a couple. Their solution of these equations was given for rectangles. In general, however, the method of solution of the equations given by them is not easily accomplished. It is the purpose of the present report to give the theory of an approximate method of solution of these equations and the results of the application of this method to tubes of rectangular, elliptical, and airfoil cross sections. This work was conducted at the Oregon State College under the sponsorship and with the financial assistance of the National Advisory Committee for Aeronautics.

SYMBOLS

In figures 1 to 5 are shown coordinate systems adopted for this report.

-	
x	coordinate along axis of tube
У¹	ordinate of airfoil measured from chord
s	distance along periphery of cross section from some arbitrary point 0'
r	normal distance from shear center of section to tangent to wall at s
\mathbf{r}_{o}	distance from leading edge to shear center
t	thickness of wall of tube
$\sigma_{\mathbf{x}}$	normal stress in skin in direction parallel to x-axis
$\sigma_{\rm S}$	normal stress in skin in direction perpendicular to $\sigma_{X}^{}$
τ	shear stress in skin in direction of x- and s-axes
$\epsilon_{\mathbf{x}}$	normal strain in skin in direction of x-axis
u	displacement of any point in skin in x-direction
θ	rate of twist of tube per unit length in x-direction
E	Young's modulus
μ	Poisson's ratio

$$I = \int tr^2 ds$$

$$I = \int tr^2 ds$$

$$I_z = \int t(y')^2 ds$$

shear modulus

$$\psi = \int_{0}^{\infty} y^{t} ds$$

$$k^2 = \frac{1 - \mu}{2}$$

a, b dimensions of rectangles and ellipses (see figs. 3 and 4)

 ξ , η coordinates of point on ellipse (see fig. 4)

$$\emptyset = \sin^{-1}\frac{\xi}{b}$$
 (for ellipse)

n, m, i positive integers

$$A_n$$
, B_n , C_n , A_m , B_m , C_m series coefficients

 λ_n coefficient

A enclosed area of cross section
$$\left(\frac{1}{2} \oint r \, ds\right)$$

total length of cross section
$$\left(\oint ds \right)$$

M twisting moment

THEORY FOR TORSION TUBE OF CONSTANT CROSS SECTION

Assumptions

The development of the equations of Karman and Chien are based on the following assumptions:

- (1) Torsion tube is of constant cross section of arbitrary shape.
- (2) Tube is loaded with constant torsional couple acting about an axis perpendicular to the cross section.
 - (3) Bending stiffness of thin walls is negligible.

- (4) Displacement of any point on wall of the tube is composed of a displacement due to rigid rotation of cross section in plane of cross section plus a displacement due to warping. The latter displacement is in a direction parallel to the axis of the cylindrical tube.
- (5) The torsional deformations can be assumed to be independent of deformations due to bending and shear loads. This implies that the principle of superposition is applicable.
- (6) Because of the presence of bulkheads, the strain in the circumferential direction is negligibly small.

General Theory

Basic equations.— On the basis of the above assumptions, Karman and Chien have shown that the following equations are applicable (reference 1, equations 7, 9, 11, and 12):.

$$\frac{2}{1-\mu}\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial s^2} + \theta \frac{dr}{ds} = 0 \tag{1}$$

$$\oint \left(\frac{\partial u}{\partial s}\right) rt ds + I\theta = \frac{M}{G}$$
(2)

$$- \oint u \frac{d\mathbf{r}}{ds} t ds + I\theta = \frac{M}{G}$$
 (2a)

from which, for constant wall thickness,

The general problem is to find a function u = u(x,s) that will satisfy equation (3) and the boundary conditions. In any particular problem r = r(s) is known from the geometry of the cross section. This may, however, not easily be expressed analytically.

An approximate solution. For the approximate solution, let the function r = r(s) be given by a trigonometric series of the following form:

$$\frac{r}{l} = C_0' + \sum_{i} C_{n'} \cos \frac{2n\pi s}{l} \qquad (4)$$

This series is periodic in the interval s = l. The coefficients may be determined by a number of methods to be discussed later.

Differentiation of equation (4) with respect to s gives

$$\frac{d\mathbf{r}}{ds} = -\sum 2n\pi \ C_n' \sin \frac{2n\pi s}{l} \tag{5}$$

Let $C_n = -2n\pi C_n!$. Then equation (5) becomes

$$\frac{d\mathbf{r}}{ds} = \sum_{n} c_n \sin \frac{2n\pi s}{l}$$
 (6)

A particular solution of equation (3) may be shown (reference 1) to be

$$u_{1}(s) = \frac{Ml}{2GAt} \left[\frac{s}{l} - \frac{l^{2}}{2A} \int_{0}^{\frac{s}{l}} \left(\frac{r}{l} \right) d\left(\frac{s}{l} \right) \right]$$
 (7)

The solution of equation (3) may then be written in the form

$$u = u_1(s) + u_2(x,s)$$
 (8)

If equations (8) and (7) are substituted in equation (3), the following is obtained:

If the condition that the end x = 0 is restrained, then at x = 0, u = 0 and $u_1 = -u_2$.

If at the other end, say $x = \infty$, the end is unrestrained, then $\sigma_{x} = 0$, or since

$$\sigma_{\rm x} = \frac{E}{1 - \mu^2}$$
 $\epsilon_{\rm x} = \frac{E}{1 - \mu^2} \frac{\partial u}{\partial x}$

then at $x = \infty$

$$\frac{\partial x}{\partial u} = \frac{\partial x}{\partial u^2} = 0$$

A function u_2 satisfying these boundary conditions may be written as

$$u_2 = -\sum A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l}$$
 (10)

The coefficients A_n may be determined by the condition that at x = 0, $u_1 = -u_2$, or

$$\sum A_{n} \sin \frac{2n\pi s}{l} = \frac{Ml}{2GAt} \left[\frac{s}{l} - \frac{l^{2}}{2A} \int_{0}^{\frac{S}{l}} \left(\frac{r}{l} \right) d\left(\frac{s}{l} \right) \right]$$
 (11)

Let

$$B_{n} = \frac{2GAt}{Ml}(A_{n}) \tag{12}$$

Equation (11) may then be written

$$\sum B_{n} \sin \frac{2n\pi s}{l} = \frac{s}{l} - \frac{l^{2}}{2A} \int_{0}^{\frac{s}{l}} \left(\frac{r}{l}\right) d\left(\frac{s}{l}\right)$$
 (13)

Equation (13) may be used to determine the coefficients B_n and consequently A_n when the geometry of the cross section is known.

The solution of equation (3) satisfying conditions at x = 0 and $x = \infty$ may then be written as

$$u = u_1(s) + u_2(x,s) = \sum A_n \sin \frac{2n\pi s}{l} - \sum A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l}$$

or

$$u = \sum A_n \left(1 - e^{-\lambda_n kx} \right) \sin \frac{2n\pi s}{l} \tag{14}$$

Since equation (14) satisfies the condition at x = 0 and $x = \infty$, it then is necessary to determine whether this is a solution that is applicable elsewhere.

The particular solution u_1 satisfies equation (3) so that it remains to determine whether u_2 satisfies equation (9). Now from equation (10)

$$\frac{1}{k^2} \frac{\partial^2 u_2}{\partial x^2} = -\sum_{n} \lambda_n^2 A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l}$$

and

$$\frac{\partial^2 u_2}{\partial s^2} = \sum \frac{\mu \pi^2 n^2}{l^2} A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l}$$

and equation (9) may, by use of the above relations and equation (10), be written

$$-\sum_{}^{}\lambda_{n}^{2}\mathrm{A}_{n}\mathrm{e}^{-\lambda_{n}kx}\,\sin\,\frac{2\mathrm{n}\pi\mathrm{s}}{l}\,+\sum_{}^{}\frac{\mathrm{L}_{1}\pi^{2}\mathrm{n}^{2}}{l^{2}}\,\mathrm{A}_{n}\mathrm{e}^{-\lambda_{n}kx}\,\sin\,\frac{2\mathrm{n}\pi\mathrm{s}}{l}\,-$$

$$\frac{t}{I} \frac{d\mathbf{r}}{ds} \oint \left(\sum_{n} A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l} \right) \frac{d\mathbf{r}}{ds} \cdot ds = 0$$
 (15)

Since dr/ds = \sum C_n sin 2n π s/l, the integral in the above equation may be evaluated. Since

$$\oint \left(\sum_{n} A_n e^{-\lambda_n kx} \sin \frac{2n\pi s}{l} \right) \left(\sum_{n} C_n \sin \frac{2n\pi s}{l} \right) ds = \frac{l}{2} \sum_{n} A_n C_n e^{-\lambda_n kx}$$

then equation (15) may be written

$$-\sum_{n}\lambda_{n}^{2}\mathbf{A}_{n}\mathrm{e}^{-\lambda_{n}kx}\,\sin\,\frac{2\mathrm{n}\pi\mathrm{s}}{\mathit{t}}+\sum_{n}\frac{\mathrm{l}\mu^{2}\mathrm{n}^{2}}{\mathit{t}^{2}}\,\mathbf{A}_{n}\mathrm{e}^{-\lambda_{n}kx}\,\sin\,\frac{2\mathrm{n}\pi\mathrm{s}}{\mathit{t}}-\mathrm{e}^{-\lambda_{n}kx}$$

$$\frac{tl}{2I} \left\langle \sum c_n \sin \frac{2n\pi s}{l} \right\rangle \left\langle \sum A_m c_m e^{-\lambda_m kx} \right\rangle = 0$$
 (16)

Equation (16) would be satisfied if every term of the summation is zero. Let any one term be denoted by the subscript i. Then

$$-\lambda_{i}^{2}A_{i}e^{-\lambda_{i}kx}\sin\frac{2\pi is}{l}+\frac{\mu^{2}i^{2}}{l^{2}}A_{i}e^{-\lambda_{i}kx}\sin\frac{2\pi is}{l}-$$

$$\frac{t}{I} C_{i} \sin \frac{2\pi i s}{l} \left(\frac{l}{2} \sum_{m} A_{m} C_{m} e^{\lambda_{m} kx} \right) = 0$$

This may be solved for λ_i as follows:

$$\lambda_{i}^{2} = \left(\frac{2\pi i}{l}\right)^{2} - \frac{tl}{2I} \cdot C_{i} \sum \frac{A_{m}C_{m}e^{-\left(\lambda_{m}-\lambda_{i}\right)kx}}{A_{i}}$$
(17)

In the development of the solution the λ_i 's were treated as constants. However, equation (17) shows that if the λ_i 's are constant,

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the basic equations are not satisfied exactly. The question then arises as to whether a value of λ_i for use in equation (10) can give a solution that approximates an exact solution to the desired degree of accuracy. Since any constant value of each λ_i will satisfy the conditions at x=0 and at $x=\infty$, it would appear that a constant λ_i can be found that will give reasonably accurate solutions in the region x>0 to $x\le\infty$.

As x approaches zero, the value of λ_i obtained from equation (17) approaches

$$\lambda_{i_0} = \sqrt{\left(\frac{2i\pi}{l}\right)^2 - \frac{tl}{2I} \frac{C_i}{A_i} \sum A_m C_m}$$
 (18)

Other values of $\lambda_{\tt i}$ were tried and were denoted by $\lambda_{\tt i_\infty}$ to distinguish from the above. These were taken as

$$\lambda_{i_{\infty}} = \sqrt{\left(\frac{2i\pi}{l}\right)^2 - \frac{tl}{2I} c_i^2}$$
 (19)

In the subsequent calculations to be discussed later, both of these sets of values were computed and used to compute the angle of twist and stresses. Good agreement with an exact solution was obtained for rectangular sections by use of $\lambda_{\bf i}$ obtained from equation (18).

If satisfactory values for λ_i are obtained, then, since the A_i 's are obtainable from equations (13) and (11), the displacement function u (equation (14)) is obtainable.

The stresses σ_X , σ_S , and τ and the angle of twist θ may be given in terms of the displacement function as follows (reference 1):

$$\sigma_{\mathbf{X}} = \frac{\mathbf{E}}{1 - \mu^2} \frac{\partial \mathbf{u}}{\partial \mathbf{x}} \tag{20}$$

$$\sigma_{\rm S} = \mu \sigma_{\rm X} \tag{21}$$

$$\theta = \frac{M}{IG} + \frac{t}{I} \oint u \frac{dr}{ds} \cdot ds$$
 (22)

$$\tau = G\left(\frac{\partial u}{\partial s} + r\theta\right) \tag{23}$$

If equation (14) is taken to represent the displacement, then the above equations become

$$\sigma_{x} = \frac{Ek}{1 - \mu^{2}} \sum_{n} \lambda_{n} A_{n} e^{-\lambda_{n} kx} \sin \frac{2n\pi s}{t}$$
 (20a)

$$\sigma_{\rm S} = \frac{E k \mu}{1 - \mu^2} \sum_{n} \lambda_n A_n e^{-\lambda_n k x} \sin \frac{2n\pi s}{t}$$
 (21a)

$$\theta = \frac{M}{IG} + \frac{t}{I} \oint \left[\sum A_n \left(1 - e^{-\lambda_n kx} \right) \sin \frac{2n\pi s}{l} \right] \times \left(\sum C_n \sin \frac{2n\pi s}{l} \right) ds \quad (22a)$$

$$\tau = G \left[\frac{2\pi}{l} \sum_{n} nA_n \left(1 - e^{-\lambda_n kx} \right) \cos \frac{2n\pi s}{l} + r\theta \right]$$
 (23a)

Assuming equation (ll₁) is a satisfactory approximation of the displacement function, then it is apparent that the stresses and angle of twist may be expressed in terms of the coefficients A_n , C_n , and λ_n which are obtainable from the geometry of the given cross section. The problem is then resolved to that of determination of these coefficients. There are several methods available for determining these.

Rectangular Cross Sections

In figure 3 is shown the notation adopted for rectangular cross sections. The shape of the rectangle may be expressed nondimensionally by the parameter

$$\beta = \frac{a}{a+b} \tag{24}$$

The coefficients C_n ' for $n \neq 0$ (equation 4) may be obtained by evaluation of the integrals

$$C_{n'} = 2 \int_{0}^{1} \frac{r}{l} \cos \frac{2n\pi s}{l} d\left(\frac{s}{l}\right)$$
 (25)

The length of the periphery of a rectangle is

$$l = 4(a + b) \tag{26}$$

The values of r are

$$0 \le s \le a \qquad r = b$$

$$a \le s \le a + 2b \qquad r = a$$

$$a + 2b \le s \le 3a + 2b \qquad r = b$$

$$3a + 2b \le s \le 3a + 4b \qquad r = a$$

$$3a + 4b \le s \le 4a + 4b \qquad r = b$$

The ratio r/l may be expressed in terms of β as follows:

$$\frac{r}{l} = \frac{b}{l} = \frac{1}{l}(1 - \beta)$$

$$\frac{s}{l} \ge 0 \le \frac{\beta}{l}$$

$$\frac{r}{l} = \frac{a}{l} = \frac{\beta}{l}$$

$$\frac{s}{l} \ge \frac{\beta}{l} \le \frac{1}{2} - \frac{\beta}{l}$$

$$\frac{r}{l} = \frac{b}{l} = \frac{1}{l}(1 - \beta)$$

$$\frac{s}{l} \ge \frac{1}{2} - \frac{\beta}{l} \le \frac{1}{2} + \frac{\beta}{l}$$

$$\frac{r}{l} = \frac{a}{l} = \frac{\beta}{l}$$

$$\frac{s}{l} \ge \frac{1}{2} + \frac{\beta}{l} \le 1 - \frac{\beta}{l}$$

$$\frac{r}{l} = \frac{b}{l} = \frac{1}{l}(1 - \beta)$$

$$\frac{s}{l} \ge 1 - \frac{\beta}{l} \le 1.00$$

The integral of equation (25) may then be evaluated to give

$$C_{n'} = \frac{1 - 2\beta}{n\pi} \sin \frac{n\pi\beta}{2} \tag{28}$$

when n is even. When n is odd, $C_n' = 0$.

The coefficients B_{n} of equation (13) may be obtained by evaluation of the integrals

$$B_{n} = 2 \int_{0}^{1} \left[\frac{s}{l} - \frac{l^{2}}{2A} \int_{0}^{\frac{s}{l}} \left(\frac{r}{l} \right) d\left(\frac{s}{l} \right) \right] \sin \frac{2n\pi s}{l} d\left(\frac{s}{l} \right)$$
 (29)

These coefficients are then

$$B_{n} = \frac{(2\beta - 1)}{(n\pi)^{2}(\beta - \beta^{2})} \sin \frac{n\pi\beta}{2}$$
 (30)

By use of the relation $C_n = -2n\pi C_n$,

$$C_n = -2n\pi C_n' = -2(1 - 2\beta) \sin \frac{n\pi\beta}{2}$$
 (31)

The angle of twist may now be determined. For a tube of circular cross section of radius b and wall thickness $t_{\rm o}$, the angle of twist is given by

$$\theta_{\text{circle}} = \frac{M}{I_{\text{o}}G}$$

where

$$I_o = t_o \oint b^2 ds = 2\pi b^3 t_o$$

If the angle of twist of a rectangle is divided by the angle of twist of a circle whose diameter is 2b and whose skin cross-section area is equal to the skin cross-section area of a rectangle,

$$\frac{\theta}{\theta_{\text{circle}}} = \frac{\theta}{\text{M/I}_0G}$$

$$= \frac{\frac{M}{IG} + \frac{t}{I} \cancel{\oint} \left[\sum_{n=1}^{\infty} A_n \left(1 - e^{-\lambda_n kx} \right) \sin \frac{2n\pi s}{I} \right] \left(\sum_{n=1}^{\infty} C_n \sin \frac{2n\pi s}{I} \right) ds}{\text{M/I}_0G}$$

and from equation (12)

$$\frac{\theta}{\text{M/I}_{o}\text{G}} = \frac{\text{I}_{o}}{\text{I}} + \frac{\imath^{2}\text{I}_{o}}{2\text{AI}} \oint \left[\sum \text{B}_{n} \left(1 - e^{-\lambda_{n}kx} \right) \sin \frac{2n\pi s}{\imath} \right] \left[\sum \text{C}_{n} \sin \frac{2n\pi s}{\imath} \right] d\left(\frac{s}{\imath} \right)$$

Also, since

$$I_{o} = 2\pi b^{3}t_{o}$$

$$i^{2} = 16(a + b)^{2}$$

$$A = 4ab$$

$$I = 4abt(a + b)$$

$$2\pi bt_{o} = 4(a + b)t$$

the above equation becomes

$$\frac{\theta}{\text{M/I}_0\text{G}} = \frac{b}{a} \left\{ 1 + \frac{2(a+b)^2}{ab} \oint \left[\sum B_n \left(1 - e^{-\lambda_n kx} \right) \sin \frac{2n\pi s}{l} \right] \left[\sum C_n \sin \frac{2n\pi s}{l} \right] d\left(\frac{s}{l} \right) \right\}$$

The integral in the expression may be determined since terms only of the form $\int_0^1 \sin^2 \frac{2n\pi s}{l} \, d \Big(\frac{s}{l} \Big)$ have values other than zero. After integration the angle of twist may be written

$$\frac{\theta}{M/I_0G} = \frac{b}{a} \left[1 + \frac{(a+b)^2}{ab} \sum_{n=0}^{\infty} B_n C_n \left(1 - e^{-\lambda_n kx} \right) \right]$$

By use of equation (24) this may be written

$$\frac{\theta}{M/I_0G} = \frac{1-\beta}{\beta} \left[1 + \frac{1}{\beta(1-\beta)} \sum_{n=0}^{\infty} \left(1 - e^{-\left(2\lambda_n b\right) \frac{kx}{2b}} \right) B_n C_n \right]$$
(32)

In a similar fashion it may be shown that the normal stress is

$$\sigma_{x} = \frac{E}{1 - \mu^{2}} \frac{Mlk}{l_{1}AbtG} \sum_{l} (2\lambda_{n}b)B_{n}e^{-(2\lambda_{n}b)\frac{kx}{2b}} \sin \frac{2n\pi s}{l}$$
(33)

If it is assumed that $A_0 = \pi b^2$, then the above equation becomes

$$\sigma_{x} = \frac{Ek}{1 - \mu^{2}} \frac{M}{A_{o}t_{o}G} \frac{A_{o}t_{o}l}{\mu Atb} \sum_{i} (2\lambda_{n}b)B_{n}e^{-(2\lambda_{n}b)\frac{kx}{2b}} \sin \frac{2n\pi s}{l}$$

A nondimensional stress $\sigma_{\mathbf{X}}^{\phantom{\mathbf{I}}}$ may be defined as

$$\sigma_{\mathbf{x}}' = \sigma_{\mathbf{x}} \frac{\left(1 - \mu^2\right) A_0 t_0 G}{EkM}$$
 (34)

For the rectangle with $l_1(a + b)t = 2\pi bt_0$, equation (34) becomes

$$\sigma_{\mathbf{x}'} = \frac{1}{2\beta(1-\beta)} \sum_{\mathbf{k}} (2\lambda_{\mathbf{n}}b) B_{\mathbf{n}}e^{-(2\lambda_{\mathbf{n}}b)\frac{\mathbf{k}\mathbf{x}}{2b}} \sin \frac{2\mathbf{n}\pi\mathbf{s}}{l}$$
(35)

A shear stress may be compared with the shear stress in a circular tube of radius b of the same skin cross-sectional area loaded with the same torsional moment. For the circular tube

$$\tau_{\rm O} = \frac{M}{2A_{\rm O}t_{\rm O}} \tag{36}$$

For the general case, from equation (23),

$$\tau = G\left(\frac{\partial u}{\partial s} + r\theta\right)$$

Using the displacement function, equation (14), and equation (12), equation (23) may be written

$$\begin{split} \tau &= G \left[\frac{\pi M}{A t G} \sum_{n \in I} n B_n \left(1 - e^{-\lambda_n k x} \right) \cos \frac{2n \pi s}{l} + r \theta \right] \\ &= \frac{M}{2 A_o t_o} \left[\frac{2\pi A_o t_o}{A t} \sum_{n \in I} n B_n \left(1 - e^{-\lambda_n k x} \right) \cos \frac{2n \pi s}{l} + \frac{2A_o t_o G l I_o}{I_o M} \left(\frac{r}{l} \right) \theta \right] \end{split}$$

Then, using equation (36), the following relation may be written

$$\frac{\tau}{\tau_{o}} = \left\{ \frac{\pi}{\beta} \sum_{nB_{n}} \left[1 - e^{-\left(2\lambda_{n}b\right)\frac{kx}{2b}} \right] \cos \frac{2n\pi s}{l} \right\} + \frac{\mu}{1 - \beta} \left(\frac{r}{l}\right) \frac{\theta}{M/I_{o}G}$$
(37)

Elliptic Cross Sections

In order to evaluate the series coefficients for elliptic cross sections, it is necessary to express r as a function of s. In figure 4 is shown the coordinate system adopted for the ellipses. With this notation,

$$s = \int_0^1 \sqrt{\frac{b^2 - (k^{\dagger})^2 \xi^2}{b^2 - \xi^2}} d\xi$$
 (38)

where

$$k' = \sqrt{\frac{b^2 - a^2}{b^2}}$$
 (39)

It may also be shown that

$$r = \frac{a}{\sqrt{1 - (k!)^2 \left(\frac{\xi}{b}\right)^2}} \tag{40}$$

Therefore

$$\frac{\mathrm{dr}}{\mathrm{ds}} = \frac{(\mathbf{k}^{\dagger})^{2} \mathbf{a}}{\mathbf{b}} \frac{\left(\frac{\mathbf{\xi}}{\mathbf{b}}\right) \sqrt{1 - \left(\frac{\mathbf{\xi}}{\mathbf{b}}\right)^{2}}}{\left[1 - (\mathbf{k}^{\dagger})^{2} \left(\frac{\mathbf{\xi}}{\mathbf{b}}\right)^{2}\right]} \tag{11}$$

Let

$$\sin \emptyset = \frac{\xi}{b} \tag{42}$$

Then equations (38), (40), and (41) may be written as

$$s = b \int_0^{\emptyset} \sqrt{1 - (k!)^2 \sin^2 \emptyset} \cdot d\emptyset$$
 (38a)

$$r = \frac{a}{\sqrt{1 - (k')^2 \sin^2 \emptyset}}$$
 (40a)

$$\frac{d\mathbf{r}}{ds} = \frac{(\mathbf{k}')^2 \mathbf{a}}{\mathbf{b}} \frac{\sin \phi \cos \phi}{\left[1 - (\mathbf{k}')^2 \sin^2 \phi\right]^2}$$
 (41a)

The integral in equation (38a) will be recognized as the elliptic integral of the second kind (reference 2) $E(\alpha,\emptyset)$. Tables of values of this integral are given in terms of the parameter \emptyset and α , where

$$\alpha = \sin^{-1} k' \tag{43}$$

The total length of the periphery of the ellipse 1 is given by

$$l = \mu b \int_0^{\frac{\pi}{2}} \sqrt{1 - (k')^2 \sin^2 \phi} \cdot d\phi \qquad (\mu \mu)$$

Since the parameter α defines the ratio of minor to major axis of an ellipse (equations (43) and (39)), this will be constant for any particular ellipse. This value is chosen first. Values of \emptyset are then assumed, from which ξ and η may be immediately computed. Values of s/b are then obtainable from the tabulated values of the elliptic functions. It is then possible to determine s/l, r/l, and dr/ds corresponding to any assumed value of \emptyset or ξ .

The coefficients $\,{}^{\mathrm{c}}_{n}\,$ for the ellipses are then determined by the equation

$$C_{n} = 2 \int_{0}^{1} \frac{d\mathbf{r}}{d\mathbf{s}} \sin \frac{2n\pi s}{l} \cdot d\left(\frac{s}{l}\right)$$
 (45)

The integrals in equation (45) may be evaluated by several methods.

In order to evaluate the coefficients $\,{\rm B}_{\rm n}$, equation (13) is used: For the ellipse this may be reduced to

$$\sum B_{n} \sin \frac{2n\pi s}{l} = \frac{s}{l} - \frac{\emptyset}{2\pi}$$
 (46)

These coefficients are then given by

$$B_{n} = 2 \int_{0}^{1} \left(\frac{s}{l} - \frac{\emptyset}{2\pi}\right) \sin \frac{2n\pi s}{l} d\left(\frac{s}{l}\right)$$
 (47)

For the elliptic thin-walled tube,

$$I = t \left(\int r^2 ds = \mu ba^2 F\left(\frac{\pi}{2}, k'\right) \right)$$
 (48)

where $F\left(\frac{\pi}{2},k^{\,\prime}\right)$ is an elliptic integral of the first kind (reference 2). Also,

$$A = \pi ab \tag{49}$$

and

$$l = 4b \left[E\left(\frac{\pi}{2}, k^{\dagger}\right) \right]$$
 (50)

$$\frac{\theta}{M/I_{o}G} = \frac{I_{o}}{I} \left\{ 1 - \frac{\pi l^{2}}{2A} \sum \left[1 - e^{-(2\lambda_{n}b)\frac{kx}{2b}} \right] nB_{n}C_{n}! \right\}$$
 (58)

$$\sigma_{\mathbf{x}'} = \frac{l^2}{8A} \sum \left[(2\lambda_n b) B_n e^{-(2\lambda_n b) \frac{k\mathbf{x}}{2b}} \sin \frac{2\pi n s}{l} \right]$$
 (59)

$$\frac{\tau}{\tau_{o}} = \left\{ \frac{\pi b l}{A} \sum_{nB_{n}} \left[1 - e^{-\left(2\lambda_{n}b\right)\frac{kx}{2b}} \right] \cos \frac{2\pi ns}{l} \right\} + \left(\frac{l}{b}\right) \left(\frac{r}{l}\right) \left(\frac{\theta}{M/I_{o}G}\right)$$
(60)

RESULTS OF CALCULATIONS

Rectangular Boxes

Calculations for angles of twist and stresses were made for rectangular boxes with cross sections defined by β = 0.05, 0.10, 0.15, 0.30, and 0.40. Computations of the coefficients C_n and B_n were first made by application of equations (30) and (31). Tabulated values of these are given in tables 1 to 6.

Values of λ_{i_0} and λ_{i_∞} were computed by use of equations (18) and (19) for values of $\beta=0.05$, 0.10, and 0.15. For $\beta=0.20$, 0.30, and 0.40, only λ_{i_0} values were computed. These values are listed in the tables in the nondimensional form $(2\lambda_n b)_0$. The angles of twist for rectangles with $\beta=0.05$, 0.10, and 0.15 are listed in table 7. These were computed from equation (32). Values were obtained based on values of both λ_0 and λ_∞ . For rectangles with $\beta=0.20$, 0.30, and 0.40, angles of twist were computed using only λ_0 . In table 7 are also given values of $\theta/(M/IG)$ based upon a value of Poisson's ratio of 0.3, giving k=0.592. This is the ratio of the unit angle of twist to that given by the theory which neglects the effect of warping of the cross section.

NACA TN 2600 19

After this shear center is located, the airfoil is plotted and values of $\rm r/\it l$ against $\rm s/\it l$ may be measured, tabulated, and plotted. The coefficients $\rm C_{\rm O}{}^{\rm I}$ and $\rm C_{\rm n}{}^{\rm I}$ are then determined from the equations

$$C_0! = \int_0^1 \left(\frac{\mathbf{r}}{l}\right) d\left(\frac{\mathbf{s}}{l}\right) \tag{54}$$

and

$$C_{i'} = 2 \int_{0}^{1} \frac{r}{l} \cos \frac{2\pi i s}{l} d\left(\frac{s}{l}\right)$$
 (55)

Equation (13) is used to evaluate the coefficients B_n . Any one coefficient B_i may be written

$$B_{i} = 2 \int_{0}^{1} \left(\frac{s}{l} \right) - \frac{l^{2}}{2A} \int_{0}^{\frac{s}{l}} \left(\frac{r}{l} \right) d\left(\frac{s}{l} \right) \sin \frac{2\pi i s}{l} d\left(\frac{s}{l} \right)$$
 (56)

Values of λ_i were computed from equation (18). This was put in the form

$$2b\lambda_{i} = \frac{\mu_{m}b_{i}}{l} \sqrt{1 - \frac{tl^{3}}{2I} \cdot \frac{C_{i}!}{iB_{i}}} \sum_{mB_{m}C_{m}!} (57)$$

by use of the relations

$$A_i = \frac{M}{2GAt} B_i$$

and

$$c_i = -2i\pi c_i$$

From equations (20a), (22a), and (23a) and by use of the relations applicable to the circular tube of same chord and skin cross-section area,

From equations (22a) and (48) to (50), it may be shown that for the elliptic cross section

$$\frac{\theta}{M/I_{o}G} = \left(\frac{b}{a}\right)^{2} \frac{E\left(\frac{\pi}{2}, k^{\dagger}\right)}{F\left(\frac{\pi}{2}, k^{\dagger}\right)} \left\{ 1 + \frac{l_{1}}{\pi} \frac{b}{a} \left[E\left(\frac{\pi}{2}, k^{\dagger}\right)\right]^{2} \sum \left[1 - e^{-\left(2\lambda_{n}bk\right)\frac{x}{2b}}\right] B_{n}C_{n} \right\}$$
(51)

Similarly, from equations (20a), (34), (49), and (50), the normal stress $\sigma_{\rm X}$ ' can be obtained as

$$\sigma_{\mathbf{x}'} = \left(\frac{\mathbf{b}}{\mathbf{a}}\right) \left(\frac{2}{\pi}\right) \left[\mathbb{E}\left(\frac{\pi}{2}, \mathbf{k'}\right)\right]^2 \sum_{\mathbf{z} \in \mathbb{Z}} 2\lambda_{\mathbf{n}} \mathbf{b} \mathbf{B}_{\mathbf{n}} e^{-\left(2\lambda_{\mathbf{n}} \mathbf{b} \mathbf{k}\right) \frac{\mathbf{x}}{2\mathbf{b}}} \sin \frac{2\mathbf{n}\pi \mathbf{s}}{l}$$
 (52)

Also, the shear stresses may be expressed as

$$\frac{\tau}{\tau_{o}} = 4 \left[\mathbb{E}\left(\frac{\pi}{2}, k!\right) \right] \left\{ \frac{b}{a} \sum_{n \in \mathbb{N}} \mathbb{E}\left[1 - e^{-\left(2\lambda_{n}bk\right)\frac{x}{2b}}\right] \cos \frac{2n\pi s}{t} + \left(\frac{r}{t}\right) \left(\frac{\theta}{M/I_{o}G}\right) \right\}$$
(53)

Airfoil-Shape Cross Section

In order to determine the coefficients C_n' and C_o' of equation (4), the contour of the airfoil box must first be expressed in the form r/l = f(s/l). In this equation r is measured from the shear center of the box. The shear center of the box was determined by the assumption that the σ_X stresses due to bending may be computed by the formula $\sigma_X = Ny'/I_Z$ where

N bending moment

y' distance from neutral axis to center of skin

 ${
m I_{Z}}$ moment of inertia of skin about neutral axis (chord line for symmetrical wing)

Results for angles of twist $\theta/(M/IG)$ are plotted in figure 6(a) for rectangles with $\beta=0.05$, 0.10, and 0.15. Values plotted were obtained using both values of λ_1 obtained from equations (18) and (19). For the rectangle $\beta=0.10$, the results obtained by Karman and Chien for the rectangle with $\beta=0.10$ are plotted for comparison. The plots are of interest from several viewpoints.

- (1) The effect of end restraint on angle of twist disappears substantially when x/2b is approximately 1.0, a distance of one chord length from the restrained end. (The chord here is considered the long side of the rectangle.)
- (2) Within small differences, the results obtained for the rectangle with $\beta = 0.10$ by Kármán and Chien check with the results obtained using the value of λ obtained from equation (18).
- (3) The series used for the computation of $\theta/(M/I_0G)$ or $\theta/(M/IG)$ converge with fair rapidity so that an excessive number of terms need not be taken.
- (4) Because the effect of the restraint is confined principally to the region near x=0, it is reasonable that the value of λ_i satisfying the basic equations in this region are the more logical ones to use.
- (5) The method of representing the contour of the box by a Fourier series gives good results for the angles of twist of rectangular cross sections.
- (6) It should be noted here that the Karman-Chien values given are not necessarily exact, as these were also obtained by the use of a finite number of terms of an infinite series.

In figure 6(b), results for angles of twist $\theta/(M/IG)$ are plotted for rectangles with β = 0.05, 0.10, 0.15, 0.20, 0.30, and 0.40. Results for angles of twist $\theta/(M/I_0G)$ for rectangles with the same β values are shown in figure 7. The values plotted in both of these figures were based on λ_0 .

The calculations for σ_x ' gave values which are listed in table 8. These are plotted in figures 8 to 13. The series for σ_x ' given in equation (35) may be shown to be divergent at the corners at x=0. This indicates an infinite stress (or the existence of a concentrated reaction) at the restrained end at the corners. Furthermore, the series converges less rapidly at the corners of the box than elsewhere. This result does not agree with the results shown in figure 5 of reference 1, in which Kármán and Chien show finite maximum stresses at the corners of

rectangular boxes. The series used by Karman and Chien may also be shown to be divergent at the corners at the restrained end. The result obtained by them was evidently obtained by use of only a finite number of terms in their series. In figures 14 and 15 are shown comparisons of values of $\sigma_{\mathbf{x}}{}^{\mathbf{t}}$ as obtained in this report with values of $\sigma_{\mathbf{x}}{}^{\mathbf{t}}$ obtained by Karman and Chien and plotted in figure 4 of reference 1.

The series for the shear stress (equation (37)) is slowly convergent, particularly near the corners of the rectangles. However, at the restrained end the shear stresses are easily computed. At the restrained end x/2b = 0,

$$\frac{\tau}{\tau_{\rm O}} = \frac{l_{\rm I}}{1 - \beta} \left(\frac{r}{l}\right) \left(\frac{\theta}{M/I_{\rm O}G}\right) \tag{61}$$

For the rectangle, since r has only two values, this may be simplified to give the following:

When r = b and x/2b = 0

$$\frac{\tau}{\tau_0} = \frac{1-\beta}{\beta} \tag{61a}$$

When r = a and x/2b = 0

$$\frac{\tau}{\tau_0} = 1.0 \tag{61b}$$

At the unrestrained end $x/2b = \infty$ it may be shown that the shear stress is given by

$$\frac{\tau}{\tau_0} = \frac{1}{2\beta} \tag{62}$$

In table 9 are shown the results obtained from calculations using the first 40 terms of the series. Because of the slow convergence the values are irregular. However, since the maximum shear values at the restrained end and the minimum values at the free end may be determined exactly, this difficulty in obtaining accurate values of shear stresses at intermediate stations was not felt to justify the extension of the computations to a larger number of terms.

It appears from the study of rectangular sections that the existence of sharp corners is a major cause of high normal and shear stresses. This suggests that the possibility of rounding corners is worthy of investigation.

23

NACA TN 2600

Boxes of Elliptical Cross Sections

Determination of coefficients.— In order to determine the basic series coefficients B_n and C_n , it is necessary to evaluate the integrals of equations (45) and (47). This was done by two methods. The first consisted of assuming values of \emptyset (equation (42)), computing values of (s/l) and dr/ds from equations (38a), (41a), and (44), and then evaluating the integrals of equations (45) and (47) by a numerical integration using a trapezoidal rule. The second method of evaluating these integrals consisted of plotting the functions $(dr/ds) \sin(2n\pi s/l)$ and $(s/l - \emptyset/2\pi) \sin(2n\pi s/l)$ against s/l and integrating graphically by the use of a planimeter. In table 10 are listed values obtained for the coefficients by these two methods for an ellipse with an a/b ratio of 0.0871.

Computation for angles of twist and stresses.— In table 11 are given values of $\theta/(M/I_0G)$ for this ellipse. Columns (1) and (2) were obtained by using coefficients C_n and B_n and λ_{n_0} obtained by calculations. Values given in column (1) were obtained by using the first 10 coefficients. Values given in column (2) were obtained by using the first 20 coefficients. Values in column (3) were obtained by using the first 10 coefficients obtained by the graphical determination.

At the unrestrained end, the simple torsion theory can be applied. This leads to a value of

$$\left(\frac{\theta}{M/I_0G}\right)_{\infty} = \frac{\mu}{\pi^2} \left(\frac{b}{a}\right)^2 \left[E\left(\frac{\pi}{2}, k^{\dagger}\right)\right]^2$$

For the ellipse with a/b = 0.0871, this equation gives $\left(\frac{\theta}{M/T_0G}\right)_{\infty}$ = 55.00. A comparison of this value with the values in table 11 shows that a 20-term series, whose coefficients were obtained by calculation, gives a very good approximation at infinity.

For the ellipse with a/b = 0.1738, calculations were based on a 20-term series using coefficients obtained by the approximate computational method of determining the integrals in equations (45) and (47). For all other ellipses calculations were based upon a 10-term series.

In tables 12 to 15 are given coefficients C_n , B_n , and $(2\lambda_n b)_o$ that were used in the computation of $\theta/(M/I_o G)$, σ_X , and τ/τ_o for the ellipses investigated.

In table 16 values are given for angles of twist for the various ellipses. These are plotted in figures 16 and 17. Both $\theta/(\text{M/I}_0\text{G})$ and $\theta/(\text{M/IG})$ are plotted.

Values of $\sigma_{\rm X}$ ' were computed using the coefficients given in columns (2), (3), and (4) of table 10 and those given in tables 12 to 15. These $\sigma_{\rm X}$ ' values are given in table 17 and are plotted in figures 18 to 22.

Values of τ/τ_0 are given in table 18.

Box of Airfoil Cross Section

The airfoil was drawn to a large scale. The coordinate system shown in figure 5 was adopted. The shear center was determined to be 36.8 units from the leading edge, the chord being taken as 65.0. The length of the periphery was obtained by summing up small distances Δs

along the contour. This length $l=\sum \Delta s=142.1$. The enclosed area of the cross section was found to be 663.0. The value of $I/t=\int r^2 ds$ was obtained by a summation of $(r^2\Delta s)$ over the periphery. This gave I/t=17,270. In table 19 are given values of r/l against s/l obtained from measurements on the airfoil.

Coefficients ${\rm C_0}^{\,\prime}$ and ${\rm C_n}^{\,\prime}$ were obtained from equations (54) and (55). The integrals were evaluated by replacing the integrals by summations. In table 20 are given the coefficients obtained. Twenty coefficients were used in the calculations. Values of ${\rm r}/l$ were then computed at various values of ${\rm s}/l$. These are listed in table 19 for comparison with the measured values. These two sets of ${\rm r}/l$ against ${\rm s}/l$ values from table 19 are plotted in figure 23. The accuracy of the series is fairly good except near the points of discontinuity of the profile.

The coefficients B_n were determined by equation (56), the integrals being replaced by summations. Values of these coefficients are listed in table 20.

Coefficients $2\lambda_n b$ were computed from equation (57). These are also listed in table 20.

In table 21 and figure 24 are given the results of computation for angles of twist. These were computed from equation (58). Angles of

twist rapidly approach a constant value as the parameter x/2b is increased. The effect of the restraint at the inner end of the box on angles of twist disappears almost entirely in one chord length from the restrained end.

The results of the calculations for the normal stresses $\sigma_{\rm X}{}^{\rm I}$ are given in table 22 and figure 25. The infinite stress at the corners of the cross section at the restrained end is again shown. In a distance of one chord length from the restrained end, however, the effect of the restraint has been reduced so that the normal stresses are negligibly small. The series used in computing $\sigma_{\rm X}{}^{\rm I}$, converges with fair rapidity so that use of 20 coefficients gives good accuracy, except right at the restrained end. Here the series converges less rapidly and is divergent at the sharp corners.

Table 23 gives the computed values of τ/τ_0 for the airfoil section. The series giving these values does not converge rapidly near the restrained end. The values given in this table are based on 20 coefficients in the series. This is insufficient for good accuracy.

CONCLUSIONS

An approximate method for solving the basic equations of Karman and Chien was determined and applied to the calculation of angles of twist and stresses in torsion boxes of rectangular, elliptical, and airfoil cross section. From the results of the application of this method, the following conclusions may be drawn:

- l. The methods given here for the approximate solution of the Karmán-Chien equations are applicable to torsion boxes having one degree of symmetry of the cross section. The series used to obtain angles of twist converge fairly rapidly. The results check well for the rectangular cross sections with those given by Karmán and Chien. The series for $\sigma_{\rm X}$ stresses converges less rapidly, particularly at the restrained end. At the corners of the restrained end, the series diverges, indicacating the necessity for concentrated loads at these points.
- 2. The theory gives results which indicate tendencies that might well be considered in design. The infinite stresses at corners at the restrained end suggest the possibility of high stress concentrations at these points. It would be expected that, even with zero circumferential strains, the computed values for angles of twist and stresses would be modified appreciably by a local yielding in the neighborhood of the restrained ends, particularly at sharp corners.

3. The studies also indicate that a wing section composed of approximately square boxes might be of some advantage in reducing angles of twist and decreasing normal stresses due to torsion.

Oregon State College Corvallis, Ore., March 28, 1950

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TABLE 1.- SERIES COEFFICIENTS FOR RECTANGLE WITH β = 0.05

L		·		
n	C _n	$\mathtt{B}_{\mathtt{n}}$	(2\lamb) _∞	$(2\lambda_n b)_0$
2 4 6 8 0 1 1 1 1 6 8 0 2 2 4 6 8 0 2 2 4 6 8 0 2 2 4 6 8 0 2 2 4 6 8 0 2 2 4 6 8 0 2 4 6 8 0 2 4 6 8 6 7 7 7 7 7 8 0	-0.281355598169 -1.0575 -1.2723 -1.4559 -1.6028 -1.7112 -1.7770 -1.7998 -1.7767 -1.7122 -1.6034 -1.4560 -1.3263 -1.0570817855622816 0 .2822 .3544 .8175 1.0581 1.2723 1.4565 1.6040 1.7127 1.7775 1.8011 1.7755 1.7122 1.5653 1.4348 1.2704 1.0581 .8179 .7875 .2792 0	-0.07507037080242101763013580107800873005860048000392003170025300111000750004600021 0 0.00017 .00031 .00049 .00058 .00058 .00058 .00058 .00058 .00058 .00049 .00039 .00049 .00039 .00034 .00039 .00039 .00039 .0003000005 000005	5.71 11.43 17.17 22.96 28.78 34.65 40.58 46.55 58.62 64.68 70.81 76.93 101.29 107.33 113.35 119.32 125.29 131.20 137.12 143.03 148.94 154.85 160.78 166.71 172.67 178.64 184.61 190.63 202.64 208.66 214.67 220.68 226.64 238.64	2.88 5.75 8.62 11.50 14.37 17.25 20.12 23.00 25.87 40.12 45.99 37.31 43.12 45.98 71.61 80.48 83.36 86.11 80.48 81.98 91.98 91.98 103.60 112.10 114.98

TABLE 2.— SERIES COEFFICIENTS FOR RECTANGLE WITH β = 0.10

n	C _n	. B _n	$(2\lambda_n b)_{\infty}$	$(2\lambda_n b)_0$
2 4 6 8 10 2 14 16 8 20 2 24 6 8 0 2 24 6 8	-0.49419397 -1.2936 -1.5208 -1.5208 -1.5208 -1.294293954940 0 .4946 .9405 1.2932 1.5210 1.5213 1.2939 .9405 .4940 049325995 -1.2913 -1.5193 -1.5193 -1.5218 -1.292093904954 0 .4945 .9405 1.2932 1.4979 1.6001 1.5238 1.2919 .9402 .4947 0	-0.069570330902024013380090600594003710020700086 0 .00057 .00092 .00108 .00109 .00100 .00084 .00063 .00011 .00019 0000160002800035000370003600031000250001800008 0 .00007 .00018 .00018 .00018 .00019	5.25 10.57 16.05 21.67 27.43 33.29 39.18 45.04 50.85 56.50 62.17 67.68 73.29 78.89 84.53 90.21 95.84 101.72 107.28 113.00 118.78 124.28 129.88 135.64 141.13 146.85 152.59 158.25 164.00 169.70 175.20 180.95 186.41 191.88 203.40 208.92 214.96 226.00	3.47 6.94 10.41 13.88 17.35 20.81 24.28 27.76 31.24 8.18 41.62 48.55 55.52 62.48 65.36 72.92 76.32 79.79 83.35 86.73 97.16 100.66 104.16 111.10 114.47 117.85 121.41 124.91 128.28 131.97 135.65 138.72

TABLE 3.- SERIES COEFFICIENTS FOR RECTANGLE WITH $\beta = 0.15$

n	C _n	B _n	$(2\lambda_n b)_{\infty}$	$(2\lambda_n b)_o$
2 4 6 8 10 11 16 8 0 2 24 6 8 0 2 3 3 3 3 4 6 8 5 5 5 5 6 6 6 6 6 6 7 7 7 7 6 8 0 6 6 6 6 6 6 6 6 7 7 7 7 8 8 0 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	-0.63554 -1.13291 -1.38248 -1.330869897343257 .21892 .82293 1.13266 1.39918 1.24758 .82142 .218804325798910 -1.33236 -1.38361 -1.326663478 0 .63566 1.13015 1.38085 1.32935 .98960 .431062204382293 -1.24570 -1.39793 -1.33048199221967 .42497 .98910 1.33387 1.40810 1.13115 .63679 0	-0.063200281501527008270040200119 .000144 .00128 .00139 .00139 .00102 .00057 .00013000220001470003500018 0 .00015 .00023 .00026 .00023 .00016 .000060000300015000150001500015000150001500016 .0000600008 .00008 .00008 .000004 .00008	4.89 9.97 15.32 20.88 26.48 31.96 37.39 42.61 47.85 53.19 58.59 69.40 74.74 79.93 85.28 90.63 95.92 101.40 106.90 112.71 128.12 133.26 138.80 144.00 149.47 154.84 160.02 165.45 170.97 176.10 181.50 186.86 191.94 197.34 202.96 208.00 213.50	3.85 7.71 11.56 15.41 19.27 23.10 27.00 30.83 34.65 540 46.20 50.10 53.96 57.75 61.55 69.30 73.22 77.17 80.92 84.75 88.65 96.23 100.20 103.95 107.92 111.82 115.58 119.47 123.44 127.13 131.02 134.92 134.92 134.13

TABLE 4.- SERIES COEFFICIENTS FOR RECTANGLE WITH β = 0.20

TABLE 5.- SERIES COEFFICIENTS FOR RECTANGLE WITH β = 0.30

n	C _n	B _n	$(2\lambda_n b)_0$
2 46 8 10 11 16 16 16 16 16 16 16 16 16 16 16 16	-0.647227608824718 .47075 .80007 .47025247067616464772 0 .646597626424655471257988247025 .24768 .76189 .64910 064885762642484447025 .80111 .47025 .80111 .47025247567596364835 0 .650237596324869462848000747025 .24630 .76365 .64659 0	-0.039070114900166 .00178 .00193 .00079000310007200049 0 .00032 .00032 .000090001500011 .00005 .00014 .00011 000009000000000600005 .0000400005 .00001000020000100002000010000200001000020000100002000010000200001000020000100002000010000200001	4.04 8.07 12.10 16.14 20.17 24.21 28.23 32.27 36.30 40.38 44.38 48.40 52.44 56.47 60.51 76.64 88.74 92.78 96.81 100.84 104.88 108.91 112.95 116.98 121.05 129.08 121.14 141.18 145.22 149.25 157.31 161.35



34

Table 6.- series coefficients for rectangle with β = 0.40

n	C _n	B _n	$(2\lambda_n b)_0$
2 4 6 8 10 11 16 18 20 22 24 26 30 31 31 34 44 44 50 52 54 56 66 66 70 72 74 76 78 80	-0.3804423412 .23512 .38032 03805723563 .23512 .38094 -03799423512 .38157 03798123487 .23512 .38182 03824523487 .23399 .37981 03788123399 .37981 03788123399 .37981 03788123399 .37981 03788123399 .37981 03788123399 .37981 03798123701 .23387 .38208 0	-0.0200700310 .00138 .00126 00005600025 .00019 .00025 00001600009 .00007 .00010 00000800004 .00004 .0000500002 .00002 .00002 .00002 .00002 .00002 .00002 .00002 .00002 .00002 .00002 .00001 .00001 .00001 .00001 .00001	3.72 7.45 11.16 14.88 18.61 22.33 26.15 29.77 33.49 37.21 40.94 44.66 48.37 52.10 55.82 59.54 63.27 66.98 70.70 74.43 78.15 81.87 85.59 89.31 93.03 96.76 100.48 107.92 111.64 115.36 119.08 122.80 126.52 130.24 133.97 145.13 148.85



TABLE 7.- ANGLES OF TWIST $\theta/(M/I_0G)$ AND $\theta/(M/IG)$ FOR RECTANGLES

(a) For rectangles with β = 0.05, 0.10, and 0.15.

	Based on λ_{∞}		Based on λ_0	
<u>x</u> 2b	θ M/I _o G	θ M/IG	$\frac{\theta}{M/I_{O}G}$	θ M∕IG
		β = 0.05		
0 1/8 1/4 1/2 3/4 1.0 2.0	19.00 81.08 89.50 94.60 95.03 95.42 95.70	1.00 4.26 4.71 4.98 5.01 5.02 5.04	19.00 68.11 81.15 89.59 92.48 93.97 95.42	1.00 3.58 4.27 4.72 4.86 4.94 5.02
		β = 0.10		_
0 1/8 1/4 1/2 3/4 1.0 2.0	9.00 19.25 21.94 23.64 24.17 24.37 24.53	1.00 2.16 2.43 2.63 2.69 2.71 2.73	9.00 17.07 20.37 22.73 23.62 24.03 24.48	1.00 1.90 2.26 2.53 2.62 2.67 2.72
β = 0.15				
0 1/8 1/4 1/2 3/4 1.0 2.0	5.67 8.79 9.68 10.48 10.76 10.89 10.98	1.00 1.55 1.71 1.85 1.90 1.92 1.94	5.67 8.23 9.30 10.24 10.62 10.80 10.98	1.00 1.45 1.64 1.81 1.87 1.91



TABLE 7.- ANGLES OF TWIST $\theta/(M/I_0G)$ AND $\theta/(M/IG)$ FOR RECTANGLES'- Concluded

(b) For rectangles with β = 0.20, 0.30, and 0.40.

	<u> </u>					
	Based or	ι λ _ο				
<u>x</u> 2b	$\frac{\theta}{M/I_{O}G}$	θ M∕IG				
	β = 0.20					
0 1/8 1/4 1/2 3/4 1.0 2.0	4.00 4.98 5.42 5.85 6.03 6.12 6.21	1.00 1.23 1.35 1.46 1.51 1.53				
	β = .0.30					
0 1/8 1/4 1/2 3/4 1.0 2.0	2.33 2.50 2.59 2.67 2.72 2.74 2.77	1.00 1.07 1.11 1.15 1.17 1.18				
	β = 0 . 40	·				
0 1/8 1/4 1/2 3/4 1.0 2.0	1.50 1.52 1.53 1.55 1.55 1.56 1.56	1.00 1.01 1.02 1.03 1.03 1.04				

Table 8.- normal stresses $\sigma_{\!_{\bf X}}{}^{\!_{1}}$ for rectangles based on $(2\lambda_n b)_{\scriptscriptstyle O}$

(a) $\beta = 0.05$.

			()	- 0						
<u>x</u> 2b	$\frac{s}{l} = 0.005$	$\frac{s}{l} = 0.01$	$\frac{s}{i} = 0.012$	$5\left \frac{s}{l}\right =$	0.02	$\frac{s}{l} =$	0.05	$\frac{s}{l} = 0.$.10	$\frac{s}{l} = 0.20$
0 1/8 1/4 1/2 3/4 1.0 2.0	-4.93 -1.99 68 18 08 04 01	-19.76 -3.62 -1.28 35 15 08 01	-4.19 -1.56 44 19	-	0.18 4.94 2.16 66 29 15 02	-2 -1 -	.17 .36 .15 .59 .33	-1.6 -1.5 -1.1 -1.0 6 1	14 10 13 13	-0.42 38 35 31 24 18 04
			(ъ)	β = 0	.10.	-			_	
<u>x</u> 2b	$\frac{s}{l} = 0.005$	$\frac{s}{l} = 0.01$	$\frac{s}{l} = 0.02$. <u>s</u> =	0.05	<u>s</u> = 0	.10	$\frac{s}{l} = 0.2$	20 -	$\frac{5}{l} = 0.025$
0 1/8 1/4 1/2 3/4 1.0 2.0	-0.69 51 23 07 03 01	-2.20 -1.01 45 14 06 03 0	-5.42 -1.83 82 26 11 06 01	-1 -1 -	.30 .84 .18 .50 .24 .13	 	92 79 49 30 18	-0.23 22 23 16 12 08		-2.08 96 31 14
		·	(c)	β = 0	.15.	· · · · · · · · · · · · · · · · · · ·	· ·	· · · · · · · · · · · · · · · · · · ·		
<u>x</u> 2b	$\frac{s}{l} = 0.02$	$\frac{s}{l} = .0.03$	$\frac{s}{l} = 0.$	0375	\frac{s}{l} =	0.05	<u>s</u> =	0.10	s l	= 0.20
0 1/8 1/4 1/2 3/4 1.0 2.0	-1.34 79 41 14 06 03	-2.16 -1.12 57 20 09 04	-1. 	65 24	-1 - - -	.95 .26 .73 .29 .13 .07		0.70 67 56 32 18 02	,	-0.15 14 15 12 08 01 01

Table 8.- Normal Stresses $\sigma_{\mathbf{x}}$ ' for rectangles Based on $\left(2\lambda_{\mathbf{n}}b\right)_{\mathbf{0}}$ - Concluded

(d) $\beta = 0.20$.

<u>x</u> 2b	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.04$	$\frac{s}{l} = 0.05$	$\frac{s}{l} = 0.06$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.20$
0 1/8 1/4 1/2 3/4 1.0 2.0	-0.54 38 23 09 04 02	-1.18 74 41 16 07 04	 -0.84 47 19	-1.28 84 50 21 11 05	-0.58 53 42 23 12 07 01	-0.12 13 12 09 06 04
			(e) β =	0.30.		
<u>x</u> 2b	$\frac{s}{l} = 0.03$	$\frac{s}{l} = 0.06$	$\frac{s}{l} = 0.075$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.15$	$\frac{s}{l} = 0.20$
0 1/8 1/4 1/2 3/4 1.0 2.0	-0.18 16 12 06 03 01	-0.49 34 22 10 05 03	-0.40 23 11 06	-0.40 33 24 12 06 03	-0.18 17 15 10 06 03	-0.08 08 07 05 03 02
			(f) β =	0.40.		
<u>x</u> 2b	$\frac{s}{l} = 0.03$	$\frac{s}{l} = 0.06$	$\frac{s}{l} = 0.10$	$\frac{s}{i} = 0.12$	$\frac{s}{l} = 0.15$	$\frac{s}{l} = 0.20$
0 1/8 1/4 1/2 3/4 1.0 2.0	-0.05 04 04 02 01 01	-0.12 09 07 04 02 01	-0.16 10 05 03	-0.16 14 10 05 03 02 0	-0.09 09 07 05 03 02	-0.04 04 03 02 02 01



Table 9.- shear stresses τ/τ_{o} for rectangles based on $\left(2\lambda_{n}b\right)_{o}$

(a) $\beta = 0.05$

	(a) $\beta = 0.05$.						
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.005$	$\frac{s}{l} = 0.01$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0 1/8 1/4 1/2 3/4 1.0 2.0	19.00 21.11 17.24 15.68 15.27 15.23 14.02	19.00 1.03 -1.61 -2.85 -3.22 -3.25 -3.56	19.00 19.18 19.92 19.40 19.15 19.15 18.87	1.00 29.32 26.34 20.77 18.19 16.89 15.30	1.00 4.79 6.72 8.70 9.45 9.68 9.61	1.00 4.08 5.28 6.71 7.65 8.31 9.42	1.00 4.05 5.26 6.59 7.51 8.19 9.42
			(b)	$\beta = 0.10.$			
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.01$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.05$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0 1/8 1/4 1/2 3/4 1.0 2.0	9.00 8.95 7.18 6.19 5.86 5.75 5.62	9.00 7.58 6.32 5.52 5.26 5.15 5.03	9.00 3.23 3.29 3.08 2.92 2.84 2.74	1.00 2.87 5.75 5.66 5.25 4.97 4.61	1.00 2.35 3.69 4.60 4.87 4.93 4.89	1.00 2.23 2.77 3.63 4.04 4.35 4.77	1.00 1.80 2.84 3.60 4.08 4.40 4.88
			(c)	β = 0.15.			
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.01$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.05$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0 1/8 1/4 1/2 3/4 1.0 2.0	5.67 5.39 4.41 3.87 3.65 3.55 3.47	5.67 5.07 4.28 3.67 3.46 3.37 3.29	5.67 5.49 4.91 4.47 4.31 4.23 4.15	1.00 3.50 3.79 3.52 3.27 3.12 2.95	1.00 1.98 2.60 3.12 3.25 3.26 3.60	1.00 1.67 2.07 2.58 2.89 3.07 3.29	1.00 1.65 2.04 2.54 2.86 3.05 3.20

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TABLE 9.- SHEAR STRESSES τ/τ_o FOR RECTANGLES BASED ON $\left(2\lambda_n b\right)_o$ - Continued

(d) $\beta = 0.20$.

			(α)	p - 0.20.			
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.01$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.05$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0	4.00	4.00	4.00	4.00	1.00	1.00	1.00
1/8	3.80	3.74	3.41	4.71 .98	1.68	1.41	1.38
1/4	3•34	3.28	2.98	4.88 .82	2.15	1.67	1.63
1/2	2.88	2.85	2.83	4.93 .54	5 • निर्म	2.01	1.97
3/4	2.70	2.68	2.46	4.90 .38	2.51	2.22	2.18
1.0	2.64	2.62	2.40	4.87 .28	2.50	2.34	2.30
2.0	2.56	2•54	2.33	4.84 .18	2.49	2.)48	2.47
			· (e)	$\beta = 0.30.$			
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.0625$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.1875$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0 1/8 1/4 1/2 3/4 1.0	2.33 2.02 1.87 1.65 1.56 1.51	2.33 2.38 2.24 2.03 1.95 1.90	2.33 2.16 2.08 2.00 1.98 1.95	1.00 1.55 1.66 1.79 1.80 1.80	1.00 1.29 1.42 1.57 1.67 1.72	1.00 1.09 1.21 1.37 1.47 1.52	1.00 1.08 1.19 1.34 1.44 1.50

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Table 9.- Shear stresses $\tau/\tau_{\rm o}$ for rectangles Based on $\left(2\lambda_{\rm n}b\right)_{\rm o}$ - Concluded

(f) $\beta = 0.40$.

		 					
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = 0.02$	$\frac{s}{l} = 0.0625$	$\frac{s}{l} = 0.10$	$\frac{s}{l} = 0.1875$	$\frac{s}{l} = 0.20$	$\frac{s}{l} = 0.25$
0	1.50	1.50	1.50	1.50 1.00	1.00	1.00	1.00
1/8	1.47	1.46	1.43	1.51	1.05	1.04	1.04
1/4	1.42	1.41	1.37	1.52 1.00	1.10	1.09	1.09
1/2	1.36	1.30	1.33	1.52 1.00	1.17	1.15	1.15
3/4	1.31	1.30	1.29	1.51 .99	1.20	1.19	1.18
1.0	1.29	1.29	1.28	1.52 1.00	1.23	1.22	1.22
2.0	1.26	1.25	1.26	1.51 .99	1:25	1.24	1.25

TABLE 10.- SERIES COEFFICIENTS FOR ELLIPSE
WITH a/b = 0.0871

(1)	(2)	(3)	(4)	(5)	(6)	(7)
n	Coefficients by calculation					
	c _n .	$B_{\mathbf{n}}$	$(2\lambda_n b)_0$	C _n	B _n	$(2\lambda_n b)_0$
2 4 6 8 10 12 14 16 18 20 21 26 28 30 32 34 36 38 40	0.25013625 .42704876 .53385725 .60326350 .58686821 .69927192 .73597477 .76017669 .77457831 .78827957	0.0467701639 .0084700550 .0039400290 .0022600181 .0014700116 .0010500092 .0008100069 .0005800048 .0005200047 .00016	4.94 10.14 11.71 19.54 24.52 29.41 34.50 39.18 44.55 53.80 58.98 64.07 68.31 74.19 78.94 83.87 87.96 93.99 97.78	0.2513844444512 .582595 .678614 .672749 .768794838851851 .838857	0.04710168 .00860063 .00360034 .00200011 .001000150010 .00090008 .00070006 .00050004 .0004	5.12 10.05 15.05 20.67 24.23 31.31 33.20 35.80 42.85 48.94

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TABLE 11.- ANGLES OF TWIST $\theta/(M/I_0G)$ AND $\theta/(M/IG)$ FOR ELLIPSE WITH a/b = 0.0871

	(1	(1))	(3)
<u>x</u> 2b	Obtained by use of first 10 coefficients of table 10, columns (2), (3), and (4)		first 10 coefficients first 20 coefficients of table 10, columns of table 10, columns		Obtained by use of first 10 coefficients of table 10, columns (5), (6), and (7)	
	e K∕I₀0	⊕ M∕IG	$\frac{\theta}{M/I_{O}G}$	θ M/IG	$\frac{\theta}{M/I_OG}$	<u>θ</u> M∕IG
0	34.90	1.00	34.90	1.00	34.90	1.00
1/8	45.02	1.29	47,29	. 1.36	44.67	1.28
1/4	48.16	1.38	50.54	1.145	48.16	1.38
1/2	50,50	1.45	52.98	1.52	50,61	1.45
3/4	51.23	1.47	53.89	1.54	51.48	1.47
1.0	51.58	1.48	54.27	1.56	51.86	1.49
2.0	52,11	1.49	54.58	1.56	52.14	1.149
ω	52.15	1.49	54.62	1.56	52.18	1.49

TABLE 12.- SERIES COEFFICIENTS FOR ELLIPSE WITH a/b = 0.1738

2 0.4378 0.04405 5.22	n	c _n	B _n	(2\lamb)0
4 5791 01457 10.39 6 .6548 .00733 15.56 8 6892 00435 20.76 10 .7223 .00291 25.86 12 7227 00201 31.38 14 .7122 .00145 36.40 16 6982 00109 39.33 18 .6848 .00085 45.38 20 6643 00068 49.38 22 .6445 .00052 57.44 24 6198 00044 63.09 26 .5944 .00036 68.46 28 5679 00028 72.89 30 .5420 .00024 78.34 32 5349 00020 82.89 34 .4858 .00016 87.93 36 4582 00012 91.54 38 .4346 .00012 99.71 40 4091 00008 100.13	4 6 8 10 12 14 16 18 20 22 24 26 28 30 32 34 36 38	5791 .6548 6892 .7223 7227 .7122 6982 .6848 6643 .6445 6198 .5944 5679 .5420 5349 .4858 4582	01457 .0073300435 .0029100201 .0014500109 .0008500068 .0005200044 .0003600028 .0002400020 .0001600012	10.39 15.56 20.76 25.86 31.38 36.40 39.33 45.38 49.38 57.44 63.09 68.46 72.89 78.34 82.89 87.93 91.54 99.71



TABLE 13.- SERIES COEFFICIENTS FOR ELLIPSE WITH a/b = 0.2588

·	
14 6439 0114 16 6 .6560 .0056 16 8 6317 0028 21 10 .5873 .0017 26 12 5401 0011 31 14 .4868 .0008 3 16 4333 0004 14 18 .3663 .0003 14	5.31 0.60 6.02 1.22 6.58 1.90 7.52 1.60 7.48 1.72



TABLE 14.- SERIES COEFFICIENTS FOR ELLIPSE WITH a/b = 0.5000

n	C _n	B_n	(2\lamble_nb)_o
2 4 6 8 10 12 14 16 18 20	0.5287 3989 .2753 1748 .0982 0643 .0386 0231 .0136 0070	0.02640 00498 .00154 00054 .00018 00009 .00004 00002	5.04 10.15 15.10 20.09 25.22 30.33 35.41 40.60 46.60 51.85



TABLE 15.- SERIES COEFFICIENTS FOR ELLIPSE WITH a/b = 0.8660

n	C _n	B _n	(2\lamb)o
2 4 6 8 10 12 14 16 18 20	0.14184 02536 .00402 00060 00098 .00008 00005 00006 .00016 00029	0.00573 0002l ₄ .00002 0 0 0 0	4.28 8.57 12.85 17.12 21.40 25.70 29.90 34.30 38.60 42.90



Table 16.- angles of Twist $\theta/(\text{M/I}_0\text{G})$ and $\theta/(\text{M/IG})$ for all ellipses from $(2\lambda_n b)_0$

x	$\frac{a}{b} = 0.0871$		$\frac{a}{b} = 0.1738$		$\frac{a}{b} = 0.2588$		$\frac{a}{b} = 0.5000$		$\frac{a}{b} = 0.8660$	
2b	$\frac{\theta}{M/I_{O}G}$	θ M/IG	$\frac{\theta}{M/I_{o}G}$	θ M/IG	$\frac{\theta}{M/I_{o}G}$	θ M/IG	$\frac{\theta}{M/I_{O}G}$	θ M/IG	$\frac{\theta}{M/I_{O}G}$	θ M/IG
0 1/8 1/4 1/2 3/4 1.0 2.0	34.90 47.29 50.54 52.98 53.89 54.27 54.58 54.62	1.00 1.36 1.45 1.52 1.54 1.56 1.56	10.97 12.88 13.59 14.16 14.38 14.48 14.55	1.00 1.17 1.24 1.29 1.31 1.32 1.33 1.33	5.80 6.36 6.61 6.84 6.93 6.96 6.99	1.00 1.10 1.14 1.18 1.20 1.20 1.21	2.25 2.34 2.40 2.46 2.49 2.51 2.51	1.00 1.04 1.07 1.09 1.11 1.11	1.16 1.16 1.16 1.16 1.16 1.16	1.00 1.00 1.00 1.00 1.00 1.00



TABLE 17.- NORMAL STRESSES $\sigma_{\mathbf{x}}$ FOR ALL ELLIPSES FROM $\left(2\lambda_{\mathbf{n}}b\right)_{\mathbf{0}}$

(a) a/b = 0.0871.

<u>x</u> 2b	$\frac{s}{7} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$					
0 1/8 1/4 1/2 3/4 1.0 2.0	0.4806 .4444 .3778 .2265 .1229 .0624 .0037	1.0609 .9727 .7444 .3872 .1915 .0936 .0052	2.5394 1.5020 .9179 .3437 .1503 .0699	3.3680 1.5068 .6832 .2084 .0842 .0385 .0020					
		(b) $a/b = 0.173$	38.						
<u>x</u> 2b	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{i} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$					
0 1/8 1/4 1/2 3/4 1.0 2.0	0.3077 .2505 .1941 .1108 .0564 .0278 .0012	0.6261 .4855 .3728 .1850 .0865 .0409 .0016	1.1263 .7468 .4363 .1632 .0675 .0302 .0012	1.5797 .6844 .3069 .0935 .0377 .0163 .0008					
		(c) $a/b = 0.258$	38.						
<u>x</u> 2b	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$					
0 1/8 1/4 1/2 3/4 1.0 2.0	0.1992 .1708 .1334 .0731 .0361 .0172 .0008	0.4154 .3339 .2462 .1198 .0549 .0253 .0012	0.7469 .4679 .2748 .1028 .0423 .0185 .0008	1.0597 .3969 .1856 .0597 .0236 .0101 .0005					

TABLE 17.- NORMAL STRESSES $\sigma_{\rm X}$ ' FOR ALL ELLIPSES FROM $\left(2\lambda_{\rm n}b\right)_{\rm O}$ - Concluded

(d) $a/b = 0.5000$	(d)	a/b	=	0.5000.
--------------------	-----	-----	---	---------

	<u> </u>								
<u>x</u> 2b	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$					
0 1/8 1/4 1/2 3/4 1.0 2.0	0.1007 .0804 .0614 .0331 .0166 .0083 .0005	0.2031 .1503 .1072 .0523 .0248 .0119 .0007	0.2700 .1633 .1013 .0420 .0186 .0086 .0005	0.221) ₄ .1088 .0617 .0238 .0102 .0047 .0003					
	(e) $a/b = 0.8660$.								
<u>x</u> 2b	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$					
0 1/8 1/4 1/2 3/4 1.0 2.0	0.0244 .0184 .0137 .0075 .0040 .0022 .0002	0.0383 .0281 .0205 .0109 .0058 .0031 .0002	0.0309 .0218 .0155 .0079 .0042 .0022 .0002	0.0175 .0122 .0086 .0043 .0023 .0012 .0001					

table 18.- shear stresses τ/τ_{o} for all ellipses from $\left(2\lambda_{n}b\right)_{o}$

(a) a/b = 0.0871

<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{4}$
0 1/8 1/4 1/2 3/4 1.0 2.0	3.04 4.29 5.04 5.97 6.48 6.74 6.98	3.14 5.16 5.96 6.87 7.30 7.50 7.69	3.56 5.36 6.20 6.89 7.09 7.15 7.19	4.66 7.68 8.24 8.09 7.88 7.76 7.64	6.35 7.90 7.33 6.34 5.94 5.76 6.72	34.80 24.61 24.53 24.91 25.16 25.54 25.34
		(b)) a/b = 0.1	L738.		
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{4}$
0 1/8 1/4 1/2 3/4 1.0 2.0	1.92 2.68 3.10 3.55 3.78 3.89 3.99	2.01 2.78 3.14 3.56 3.77 3.86 3.94	2.28 2.97 3.35 3.66 3.75 3.79 3.80	3.12 4.26 4.48 4.39 4.30 4.25 4.21	4.38 6.25 5.99 5.63 5.49 5.44 5.39	10.97 6.32 5.67 5.36 5.29 5.27 4.87
		(c)	a/b = 0.2	2588.		
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{l_4}$
0 1/8 1/4 1/2 3/4 1.0 2.0	1.47 1.85 2.09 2.23 2.53 2.60 2.66	1.52 1.90 2.12 2.39 2.51 2.57 2.61	1.75 2.20 2.41 2.60 2.65 2.66 2.67	2.15 2.51 2.56 2.47 2.41 2.37 2.35	3.40 3.52 3.27 3.01 2.90 2.86 2.83	5.69 3.86 3.38 3.94 3.00 2.96 2.93



TABLE 18.- SHEAR STRESSES τ/τ_0 FOR ALL ELLIPSES FROM $(2\lambda_n b)_0$ - Concluded (d) a/b = 0.5000.

<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{i} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s.}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{4}$				
0 1/8 1/4 1/2 3/4 1.0 2.0	1.12		1.30		1.98 1.88 1.80 1.71 1.69 1.67	2.23 1.96 1.84 1.74 1.70 1.68 1.67				
	(e) $a/b = 0.8660$.									
<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{4}$				
0 1/8 1/4 1/2 3/4 1.0 2.0	1.00 1.02 1.04 1.06 1.07 1.07	1.02 1.04 1.05 1.06 1.07 1.07	1.07 1.07 1.07 1.07 1.07 1.07	1.13 1.12 1.11 1.09 1.09 1.08 1.08	1.14	1.16 1.13 1.12 1.10 1.08 1.08				

TABLE 19.- COORDINATES OF NACA 631-012 AIRFOIL SECTION TORSION BOX

<u>s</u> 1	$\frac{\mathbf{r}}{l}$	<u>s</u> 1	$\frac{r}{l}$ (1)
0 .015 .028 .047 .065 .083 .119 .155 .190 .225 .261 .296 .332 .367 .1402 .437 .473	0.259 .141 .105 .084 .065 .061 .050 .049 .046 .043 .042 .042 .042 .045 .045 .045	0 .025 .050 .075 .100 .125 .150 .175 .200 .225 .250 .275 .300 .325 .350 .375 .400 .125 .1450 .1475	0.2244 .1138 .0689 .0582 .0488 .0487 .0487 .0511 .0477 .0476 .0494 .0466 .0457 .0485 .0447 .0440 .0161 .0430 .0385 .1512 .2160

 $\frac{1}{\overline{t}} \text{ computed from } \frac{r}{\overline{t}} = C_0' + \sum_{n'} C_n' \cos \frac{2\pi ns}{\overline{t}}$

TABLE 20.- SERIES COEFFICIENTS FOR NACA 631-012 AIRFOIL SECTION

n ·	c _n '	$^{\cdot}$ $^{B}_{n}$,	(2\(\lambda_n\)b)
1 2 3 4 5 6 7 8 9 0 11 12 13 14 15 16 17 18 19 20	0.00288 .03445 .00205 .03068 .03068 .02553 .00409 .02054 .01613 .00613 .00185 .00588 .00377 .00351 .00549 .00183 .00668 .00038	-0.0066001886 .0020601870 .0019001033 .0016000598 .0005300367 .00030002370004500134000610005600093000930000600006	2.31 4.86 7.38 9.41 11.79 14.10 16.90 18.61 21.31 23.12 26.46 27.92 32.25 33.79 35.26 38.06 40.98 43.03 44.76 48.01



TABLE 21.- ANGLES OF TWIST $\theta/(\text{M/I}_0\text{G})$ AND $\theta/(\text{M/IG})$ FOR NACA 63_1 -O12 AIRFOIL SECTION

<u>x</u> 2b	θ M/I _o G	θ M/IG
0	8.70	1.00
1/4	11.64	1.34
1/2	12.33	1.42
1.0	12.61	1,45
1.5	12.67	1.46
2.0	12.69	1.46
4.0	12.69	1.46



TABLE 22.- SUMMARY OF NORMAL STRESSES $\sigma_{\mathbf{x}}{}^{\mathbf{1}}$ FOR NACA 631-012 AIRFOIL SECTION

<u>x</u> 2b	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{l}$	$\frac{s}{7} = \frac{3}{8}$	$\frac{s}{l} = \frac{7}{16}$	$\frac{s}{l} = \frac{15}{32}$
0 1/4 1/2 1.0 1.5 2.0 4.0	-1.90 -1.00 36 09 02 01	-1.20 82 39 12 03 01	-0.83 47 25 09 03 01 0	-0.18 40 16 07 02 01	-0.48 11 07 03 01 01	0.83 .70 .32 .08 .01 0	2.38 1.07 .35 .07 .01 0	5.13 .72 .22 .04 .01 0

TABLE 23.- SUMMARY OF SHEAR STRESSES τ/τ_{O} FOR

NACA 631-012 AIRFOIL SECTION

<u>x</u> 2b	$\frac{s}{l} = 0$	$\frac{s}{l} = \frac{1}{16}$	$\frac{s}{l} = \frac{1}{8}$	$\frac{s}{l} = \frac{3}{16}$	$\frac{s}{l} = \frac{7}{32}$	$\frac{s}{l} = \frac{1}{l_4}$	$\frac{s}{l} = \frac{3}{8}$	$\frac{s}{l} = \frac{7}{16}$	$\frac{s}{l} = \frac{15}{32}$	$\frac{s}{l} = \frac{1}{2}$
0 1/4 1/2 1.0 1.5 2.0 4.0	6.14	2.62 3.95 1.80 1.64 1.57 1.56 1.55	1.90 2.26 3.18 3.30 3.31 3.30 3.29	3.50 3.52	1.63 2.23 3.11 3.38 3.46 3.48 3.49	1.60 2.68 3.21 3.49 3.60 3.62 3.63	1.71 3.77 3.63 3.82 3.85 3.87 3.88	4.21 4.09 4.05	1.90 .04 .03 24 31 32 32	7.53 2.91 3.10 2.92 2.89 2.89 2.89

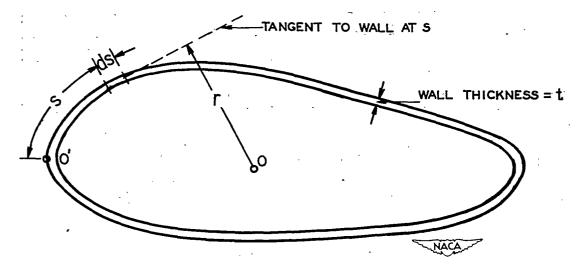


Figure 1.- Cross section of torsion tube.

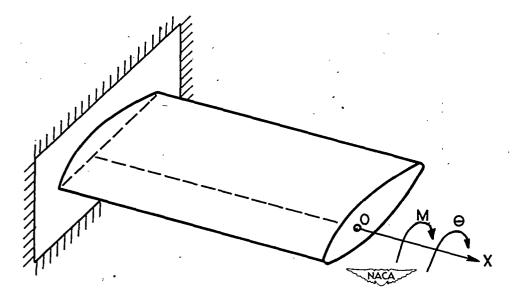


Figure 2.- Perspective of torsion tube.

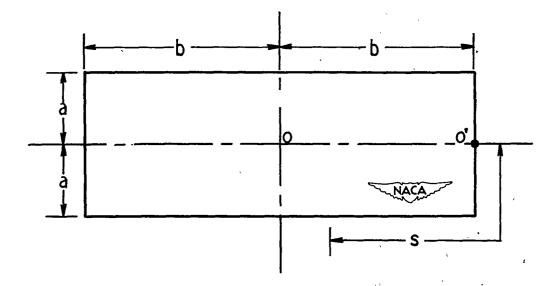


Figure 3.- Notation for rectangular box.

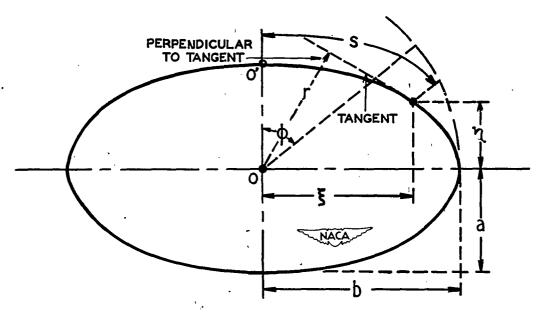


Figure 4.- Notation for elliptical box.

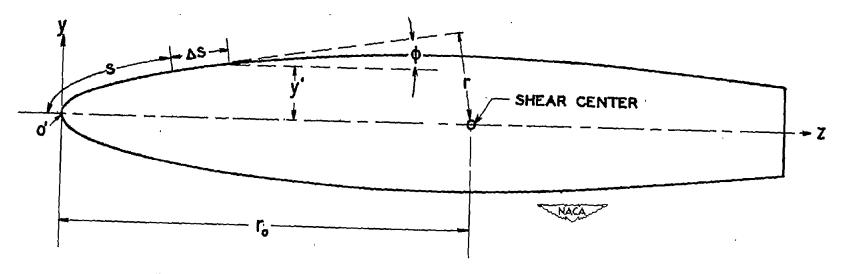
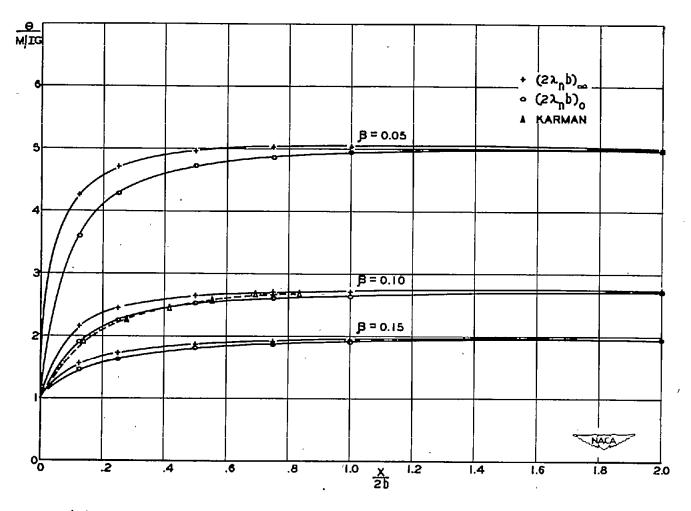
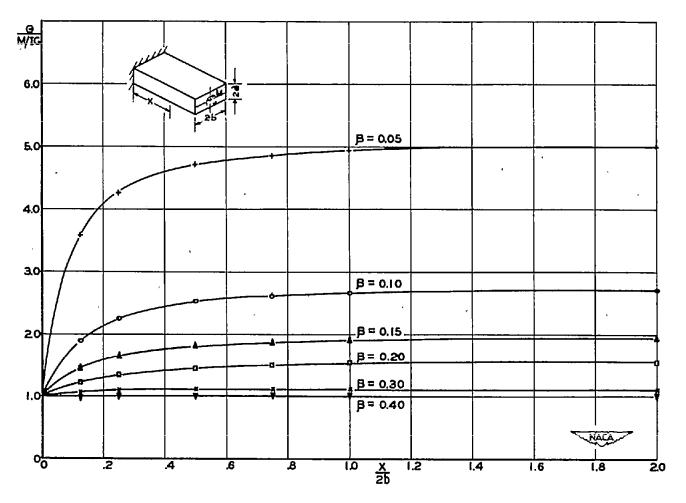


Figure 5.- Torsion tube 65 percent of NACA 631-012 airfoil section.



(a) Values based on λ_0 and λ_∞ . Comparison with values obtained by Karman and Chien (reference 1) is shown for $\beta=0.10$.

Figure 6.- Variation of angles of twist $\theta(M/IG)$ with x/2b for rectangular boxes with various β values.



(b) Values based on λ_0 .

Figure 6.- Concluded.

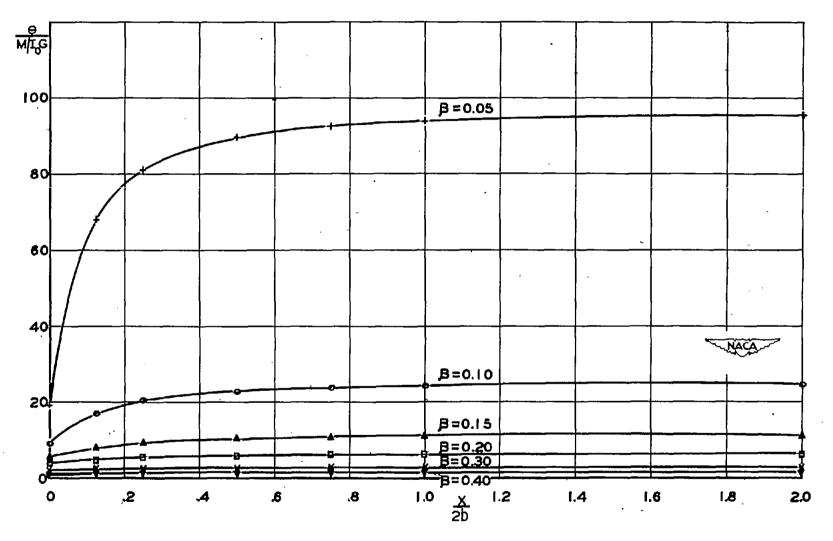


Figure 7.- Variation of angles of twist $\theta/(MI_0G)$ with x/2b for rectangular boxes with various β values and based on λ_0 .

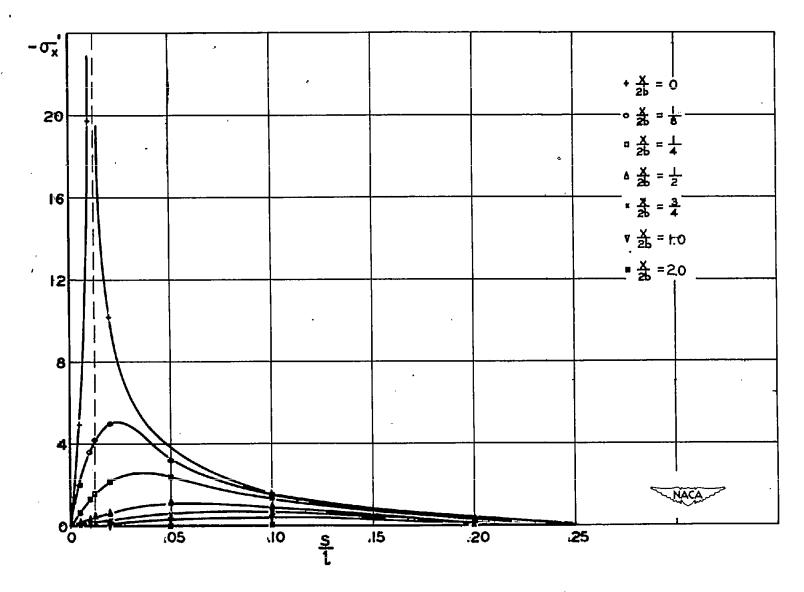


Figure 8.- Variation of normal stresses $-\sigma_{\overline{x}}'$ with s/l for rectangular boxes with $\beta=0.05$.

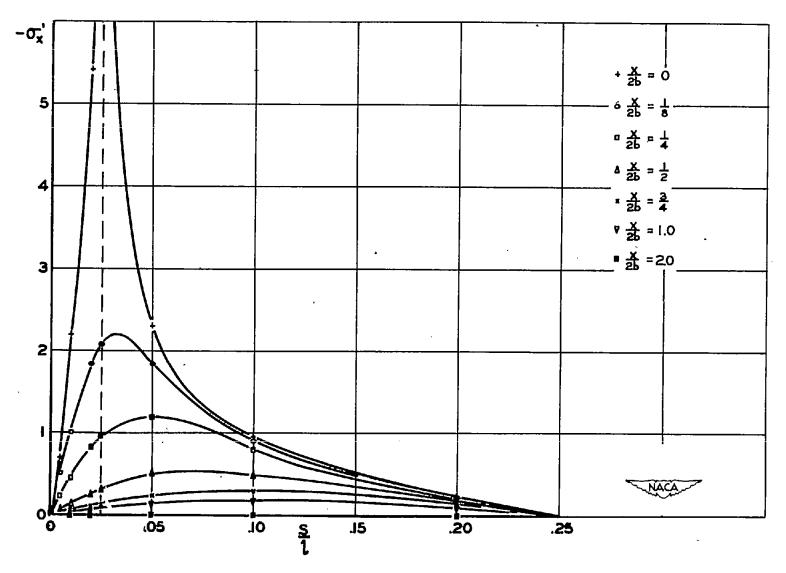


Figure 9.- Variation of normal stresses $-\sigma_{\mathbf{X}}$ with s/l for rectangular boxes with $\beta=0.10$.

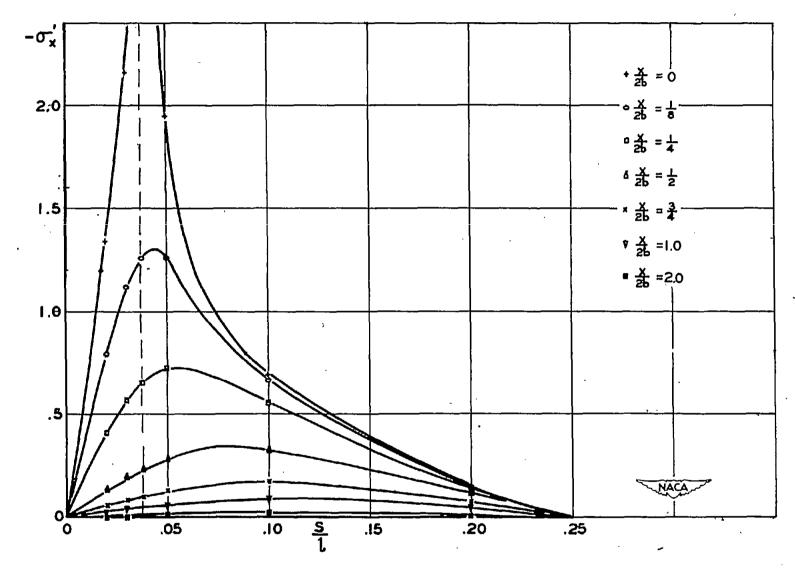


Figure 10.- Variation of normal stresses $-\sigma_{\rm x}$ ' with s/l for rectangular boxes with β = 0.15.

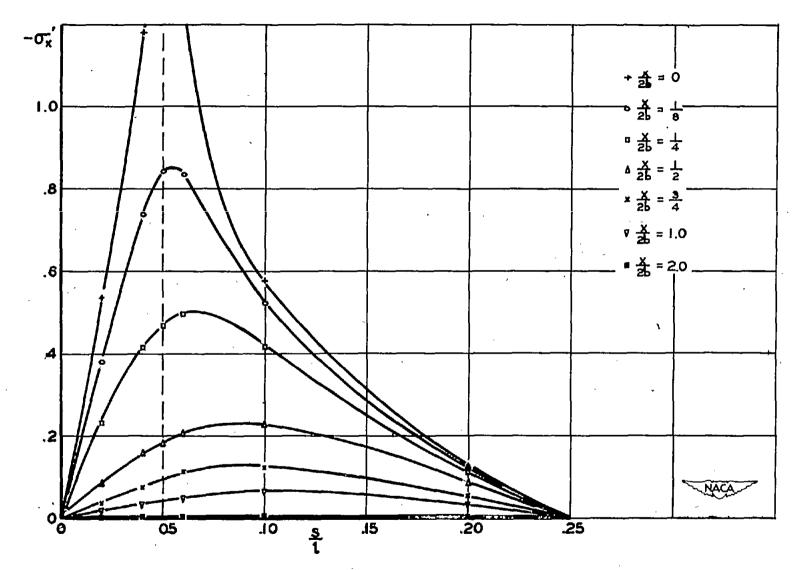


Figure 11.- Variation of normal stresses $-\sigma_X$ ' with s/t for rectangular boxes with β = 0.20.

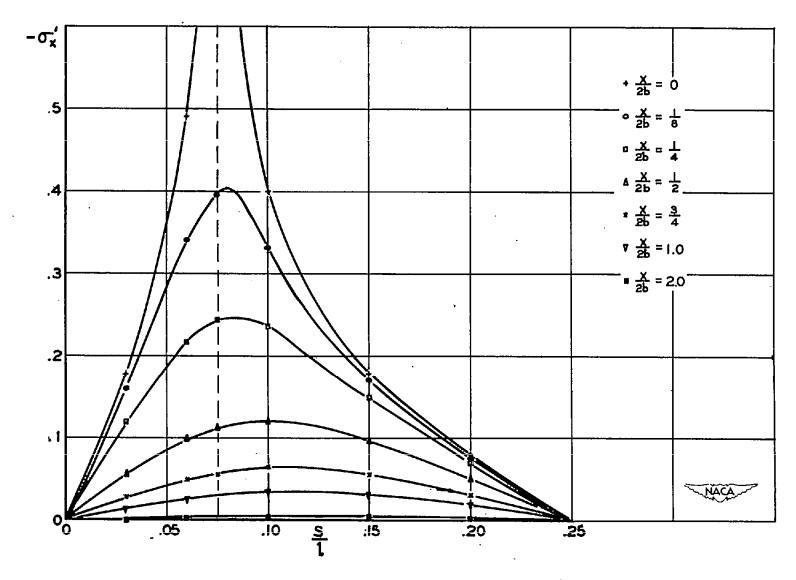


Figure 12.- Variation of normal stresses $-\sigma_{\mathbf{x}}^{\phantom{\mathbf{x}}\prime}$ with s/l for rectangular boxes with $\beta = 0.30$.

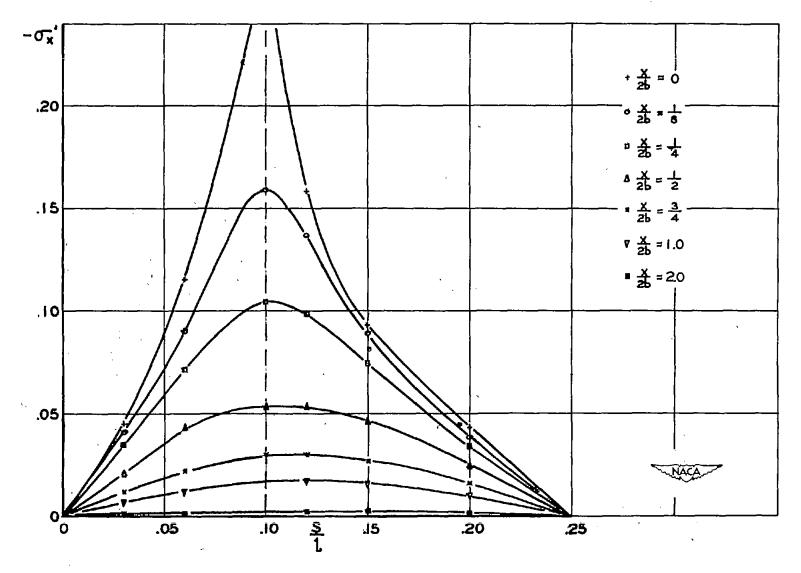


Figure 13.- Variation of normal stresses $-\sigma_{\bf x}$ with s/l for rectangular boxes with $\beta=0.40$.

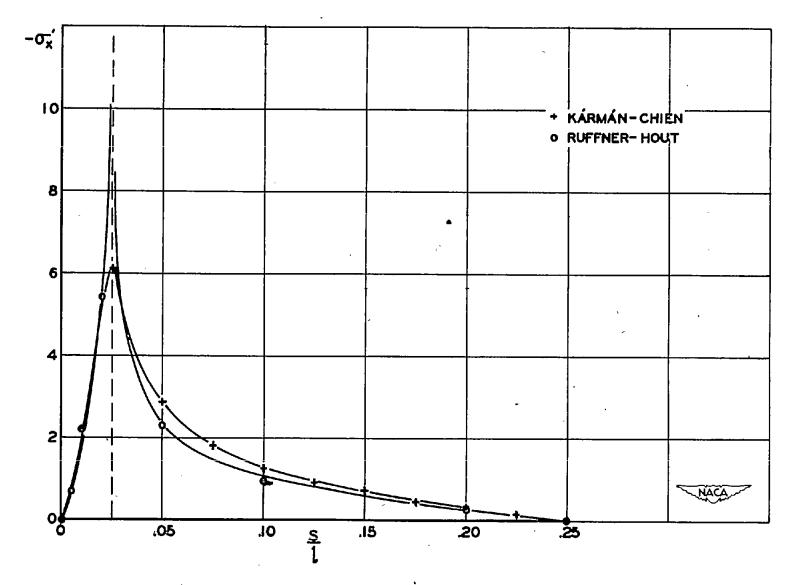


Figure 14.- Comparison of normal stresses for rectangular box with $\beta=0.1$. $-\sigma_{\rm X}$ at ${\rm x}/2b=0$.

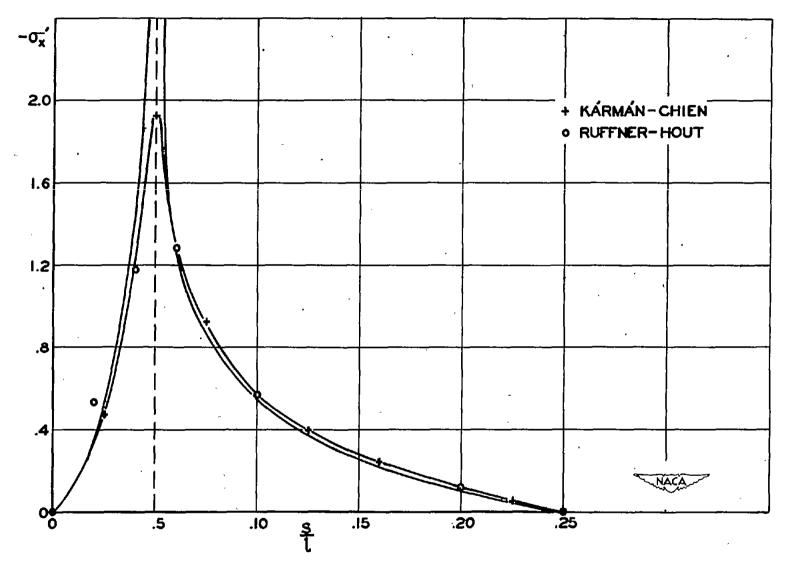


Figure 15.- Comparison of normal stresses for rectangular box with β = 0.2. $-\sigma_{\rm x}^{\ i}$ at ${\rm x}/{\rm 2b}$ = 0.

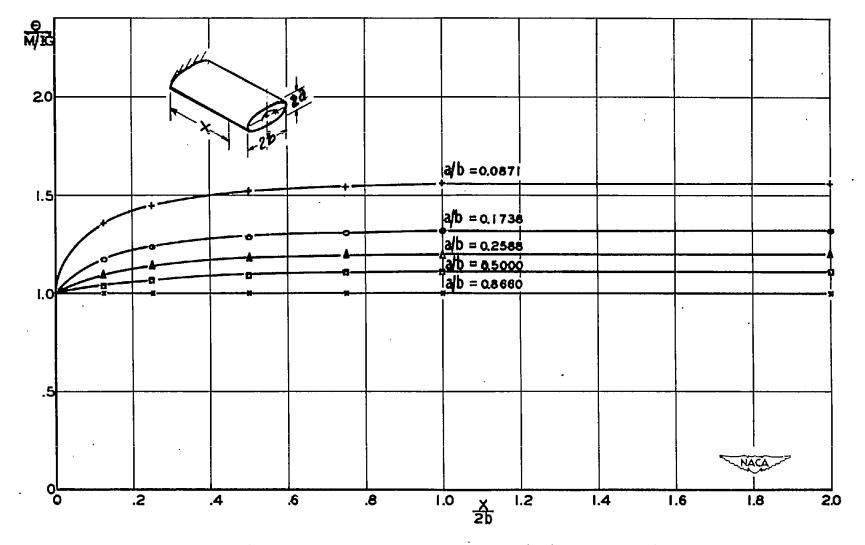


Figure 16.- Variation of angles of twist $\theta/(M/IG)$ with x/2b for elliptical boxes

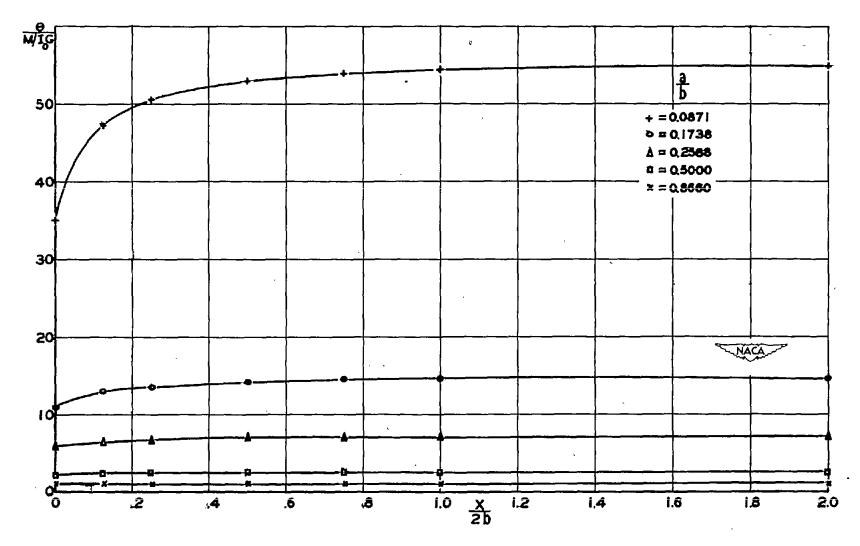


Figure 17.- Variation of angles of twist $\theta/(M/I_0G)$ with x/2b for elliptical boxes.

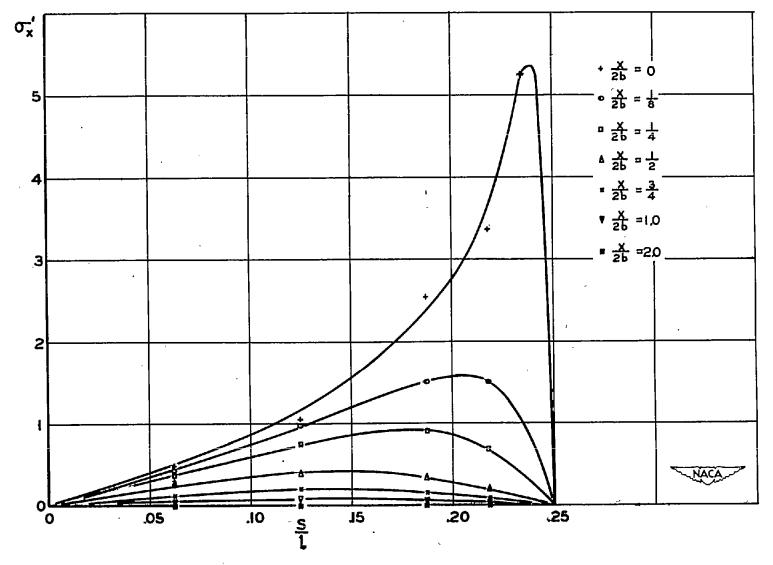


Figure 18.- Variation of normal stresses σ_{x} with s/l for elliptical boxes with a/b = 0.0871.

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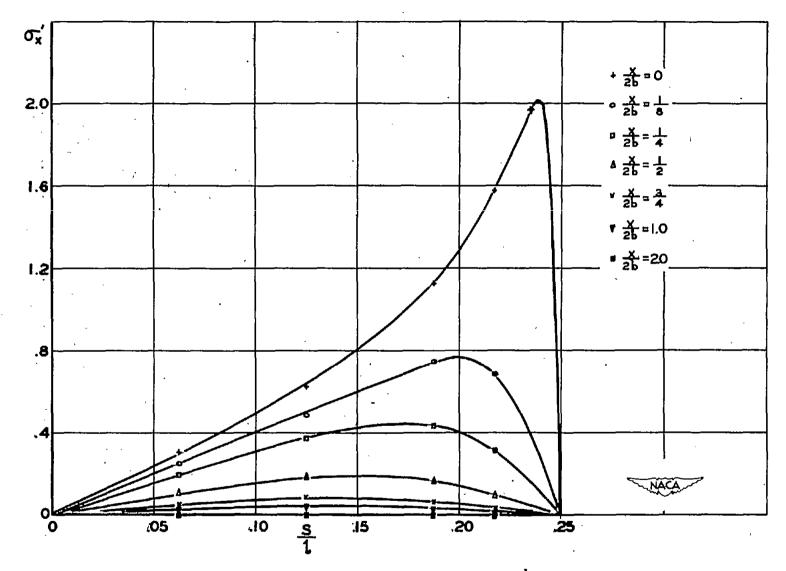


Figure 19.- Variation of normal stresses σ_{x} with s/l for elliptical boxes with a/b = 0.1738.

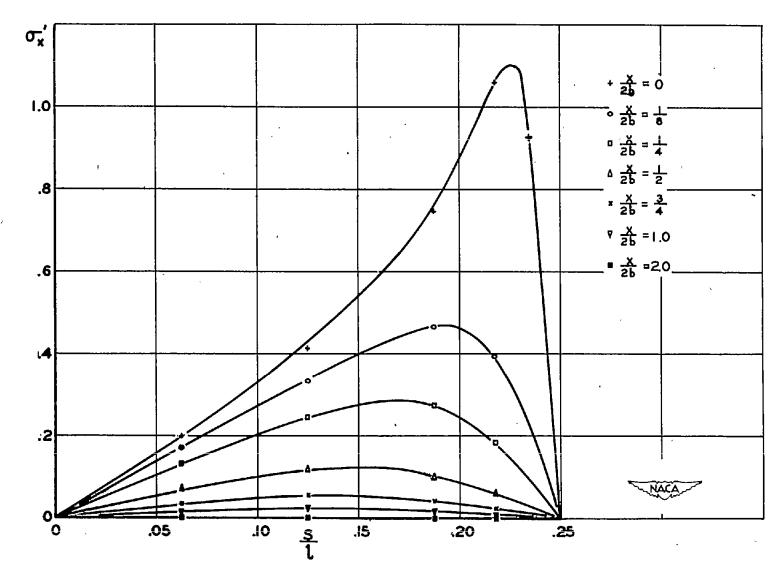


Figure 20.- Variation of normal stresses $\sigma_{\rm X}^{\ t}$ with s/l for elliptical boxes with a/b = 0.2588.

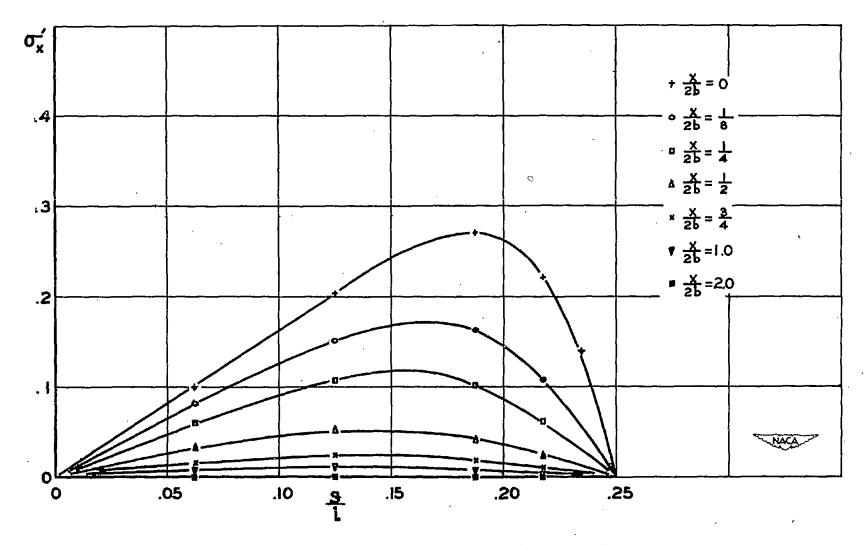


Figure 21.- Variation of normal stresses $\sigma_{\bf x}'$ with s/l for elliptical boxes with a/b = 0.5000.

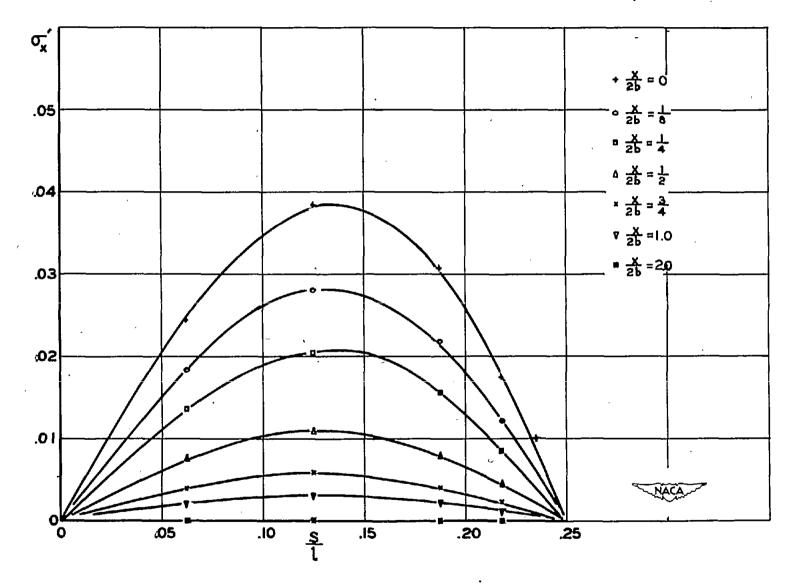


Figure 22.- Variation of normal stresses $\sigma_{\rm x}$ with s/l for elliptical boxes with a/b = 0.8660.

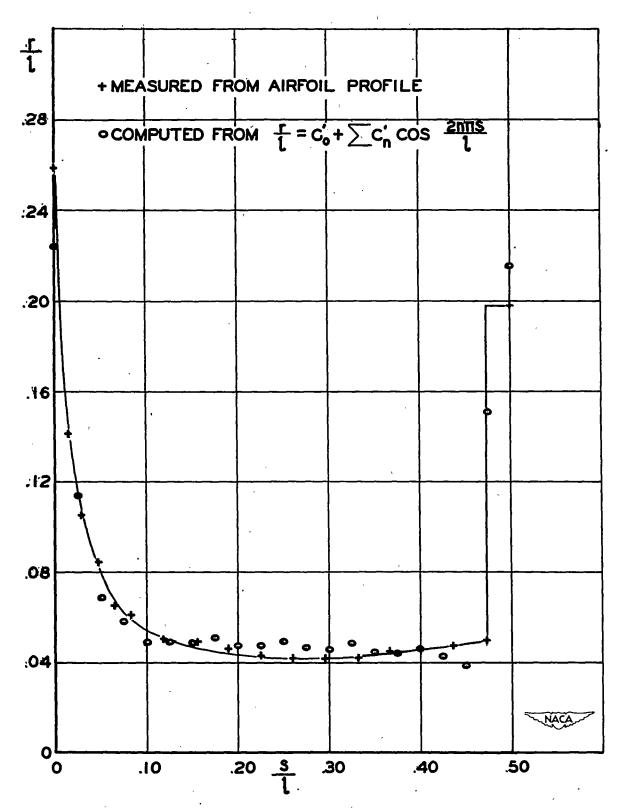


Figure 23.- Contour of NACA 631-012 airfoil.

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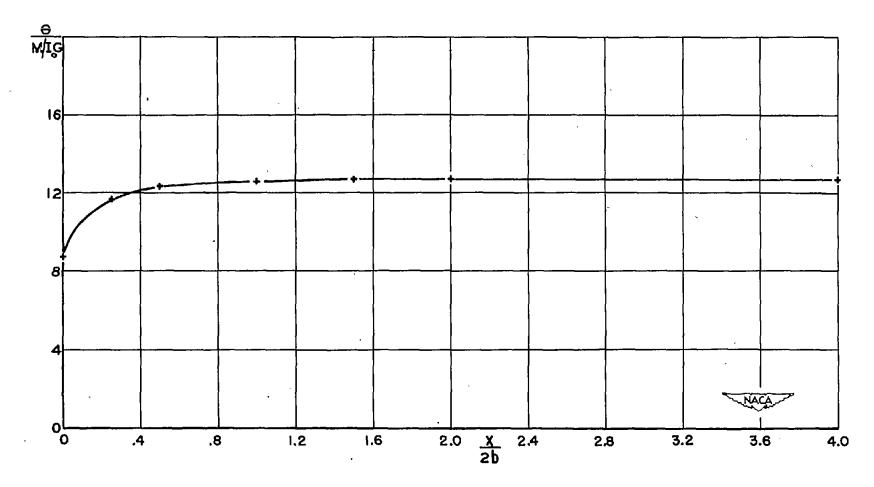


Figure 24.- Variation of angles of twist $\theta/(M/I_0G)$ with x/2b for NACA 63_1 -012 airfoil section.

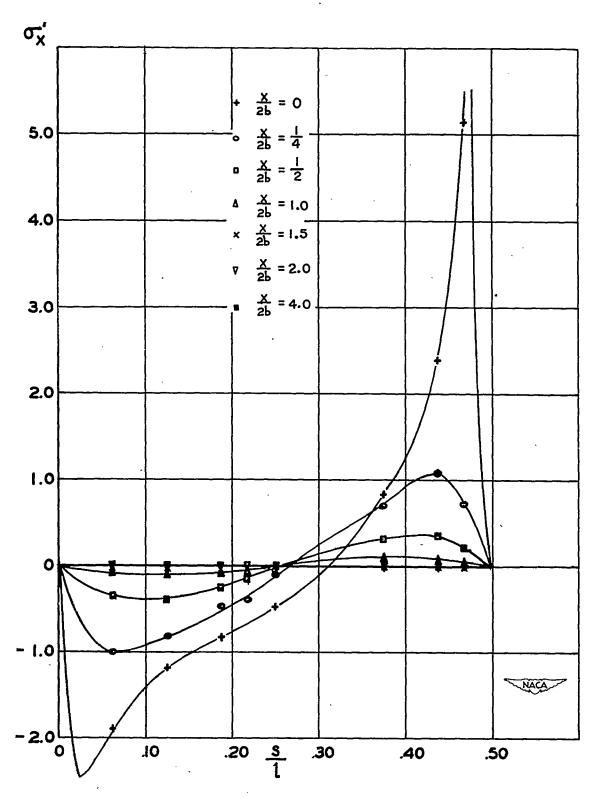


Figure 25.- Variation of normal stresses $\sigma_{\mathbf{x}}{}^{!}$ with s/l for NACA 631-012 airfoil.